



Applications of *MAT_258

A through-thickness regularization model for shells

DYNAMore NORDIC USERS' CONFERENCE
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Tore Børvik, Odd Hopperstad, Magnus Langseth...

Outline

1. A through-thickness regularization model for shells
2. Executive summary of the model formulation
3. Calibration using one **uniaxial tension** test
4. Effects of **regularization**
5. Validation using **three-point bending** and **crushing** tests
6. What about **coarser meshes**?
7. Summary

1

INTRODUCTION

Introduction

Available in LS-DYNA starting with version R9.3

***MAT_258**

***MAT_NON_QUADRATIC_FAILURE**

***MAT_NON_QUADRATIC_FAILURE**

This is material 258. This is an elastic-(visco)plastic material with a non-quadratic yield surface where isotropic work hardening is included. A ductile failure model is included in the form of a damage indicator model. The extended Cockcroft-Latham criterion is

Basic idea:

1. To compute failure as a function of the bending-to-membrane loading ratio of each element
2. To reduce the mesh dependency of the failure strain under membrane loading
3. To have a conservative but accurate model that requires few tests to calibrate

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MODEL FORMULATION

Constitutive model

- Hershey-Hosford yield criterion for isotropic plasticity:

$$f = \bar{\sigma} - (\sigma_Y + R(\bar{\varepsilon}^p)) \quad \bar{\sigma} = \left[\frac{1}{2} (|\sigma_1 - \sigma_2|^a + |\sigma_2 - \sigma_3|^a + |\sigma_3 - \sigma_1|^a) \right]^{\frac{1}{a}}$$

- Three-term voce hardening rule (7 parameters):

$$R(\bar{\varepsilon}^p) = \sum_{i=1}^3 Q_i \left(1 - \exp \left(-\frac{\theta_i}{Q_i} \bar{\varepsilon}^p \right) \right)$$

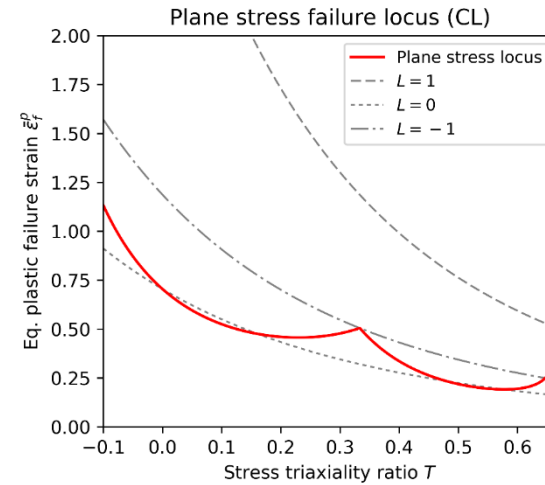
- Viscoplasticity is available:

$$\bar{\sigma} = (\sigma_Y + R(\bar{\varepsilon}^p)) \left(1 + \frac{\dot{\bar{\varepsilon}}^p}{\dot{\bar{\varepsilon}}_\sigma^p} \right)^{C_\sigma} \text{ for } f > 0$$

Failure model

- Uncoupled version of the **Extended Cockcroft-Latham (ECL)** criterion:

$$\dot{D} = \frac{\bar{\sigma}}{W_C} \left\langle \phi \frac{\sigma_I}{\bar{\sigma}} + (1 - \phi) \left(\frac{\sigma_I - \sigma_{III}}{\bar{\sigma}} \right) \right\rangle^\gamma \dot{\varepsilon}^p$$

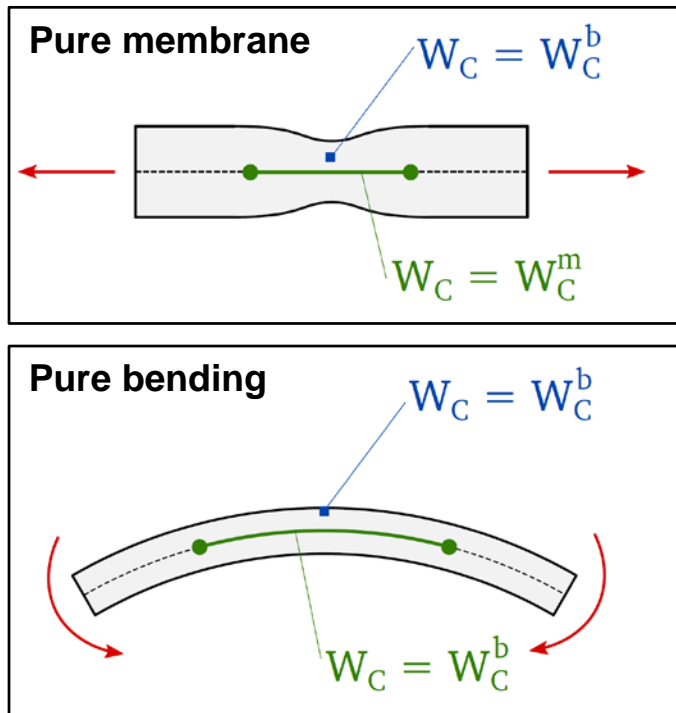


- The model parameters W_C , ϕ , and γ govern the damage evolution and its dependence on stress triaxiality and Lode parameter
 - W_C determines the overall level of the failure strain and is used in the regularization

Gruben G, Hopperstad OS, Børvik T. *Evaluation of uncoupled ductile fracture criteria for the dual-phase steel Docol 600DL*. International Journal of Mechanical Sciences 2012; 62: 133-146.

Regularization procedure

- The **bending-to-membrane loading ratio** Ω is computed using the through-thickness variation of the through-thickness plastic strain
- W_C^l, W_C^s , and c determine the **element size regularization**



$$W_C = \Omega W_C^b + (1 - \Omega) W_C^m$$

$$W_C^b = W_C^b$$

$$W_C^m = W_C^l + (W_C^s - W_C^l) \exp\left(-c \left(\frac{l_e}{t_e} - 1\right)\right)$$

$$\Omega = \frac{1}{2} \frac{|\varepsilon_{33p}^T - \varepsilon_{33p}^B|}{|\max\{\varepsilon_{33p}^T, \varepsilon_{33p}^B\}|}$$

Costas M, Morin D, Hopperstad OS, Børvik T, Langseth M. A through-thickness damage regularization scheme for shell elements subjected to severe bending and membrane deformations. Journal of the Mechanics and Physics of Solids. In press.

3

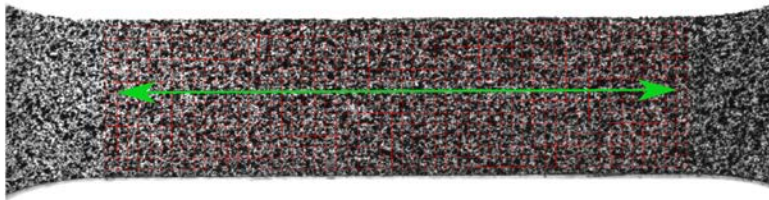
CALIBRATION USING A UNIAXIAL TENSION TEST

Calibration procedure

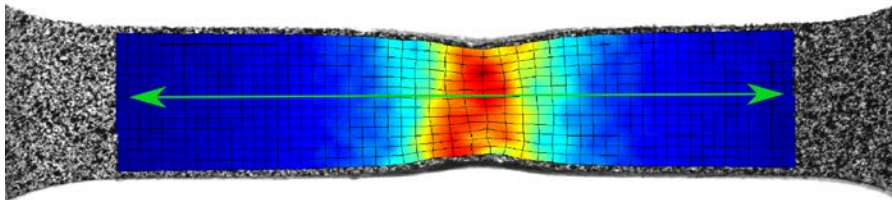
1. Conduct tension test
2. Simulate tension test with solid elements
 - Determine the hardening parameters using inverse modeling techniques (σ_Y , θ_1 , C_1 , θ_2 , C_2 , θ_3 , C_3)
 - Integrate the major principal stress over the equivalent plastic strain until fracture to find the “true/bending” CL parameter W_C^b
3. Stretch single shell elements of various sizes using elongation from DIC virtual extensometers of corresponding length as boundary conditions
 - Integrate the major principal stress over the equivalent plastic strain until fracture to find the membrane CL parameters (W_C^m) for different element sizes and thus W_C^l , W_C^s , and c

Tension testing

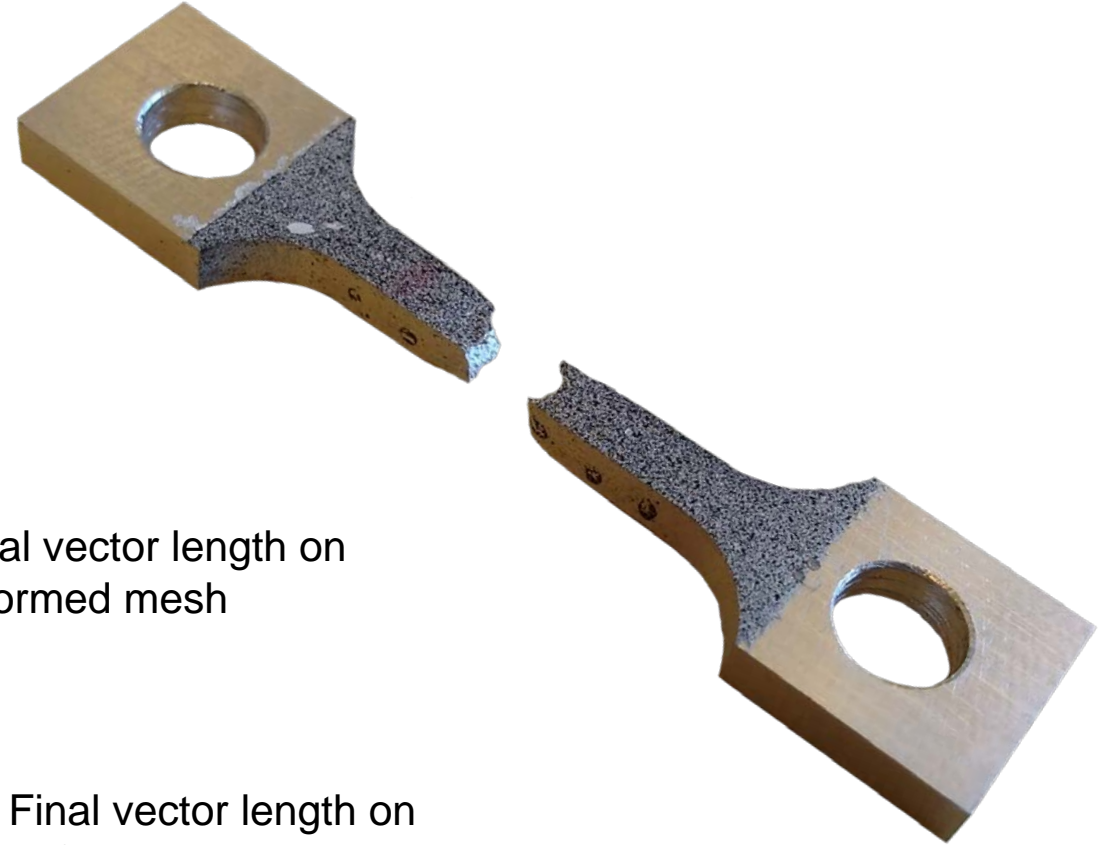
- UT specimens : 3 mm wide in gauge area
- Crosshead velocity : 0.67 mm/min
- Test duration \approx 4 minutes
- DIC : 2 fps
- Vector length \approx 10 mm



Original vector length on undeformed mesh



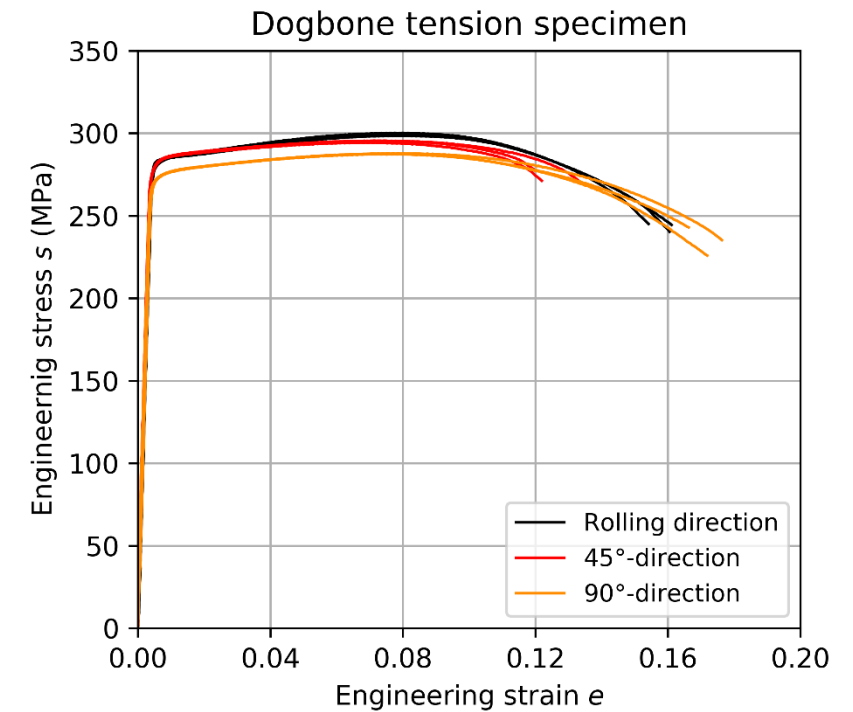
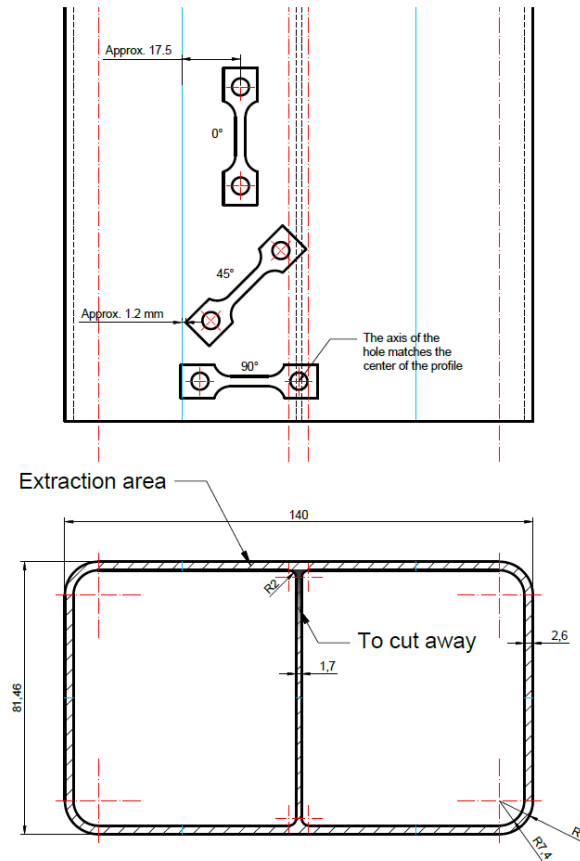
Final vector length on deformed mesh



Tension testing

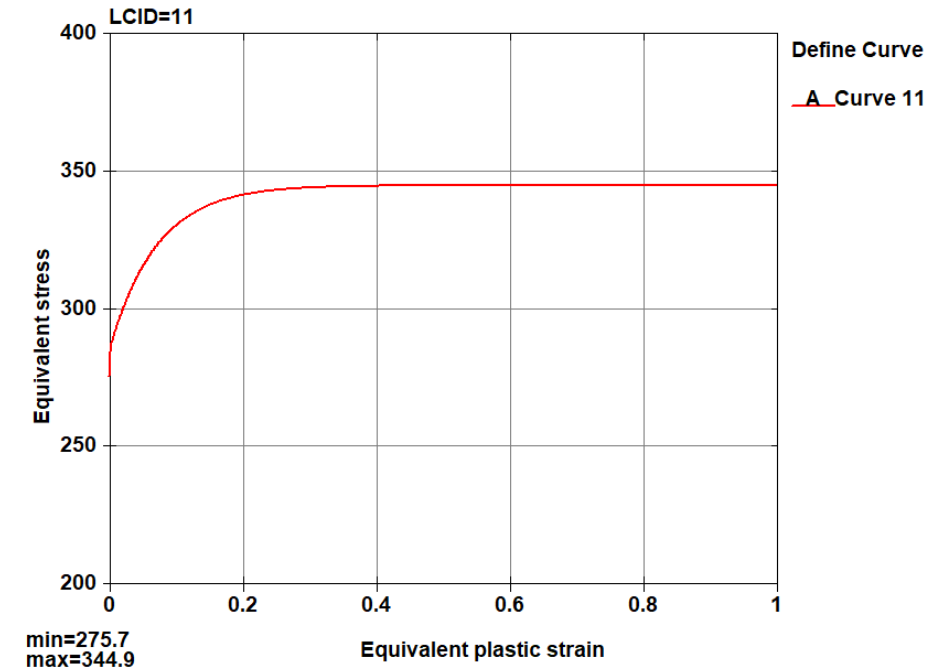
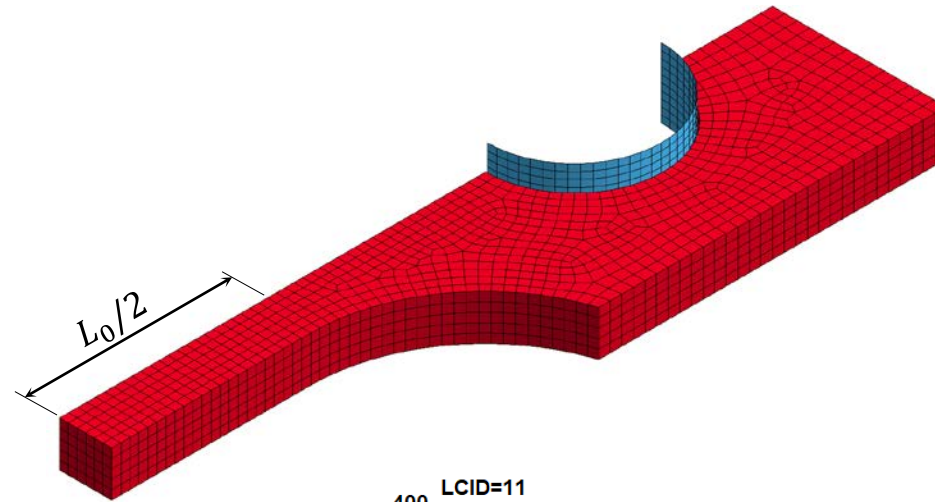
AA6005-T6

- Extracted specimens from the base material in the outer wall
- Three repetitions per orientation
- $s = F/A_0$, $e = \Delta L/L_0$
- **One test from the extrusion direction** is used in the following



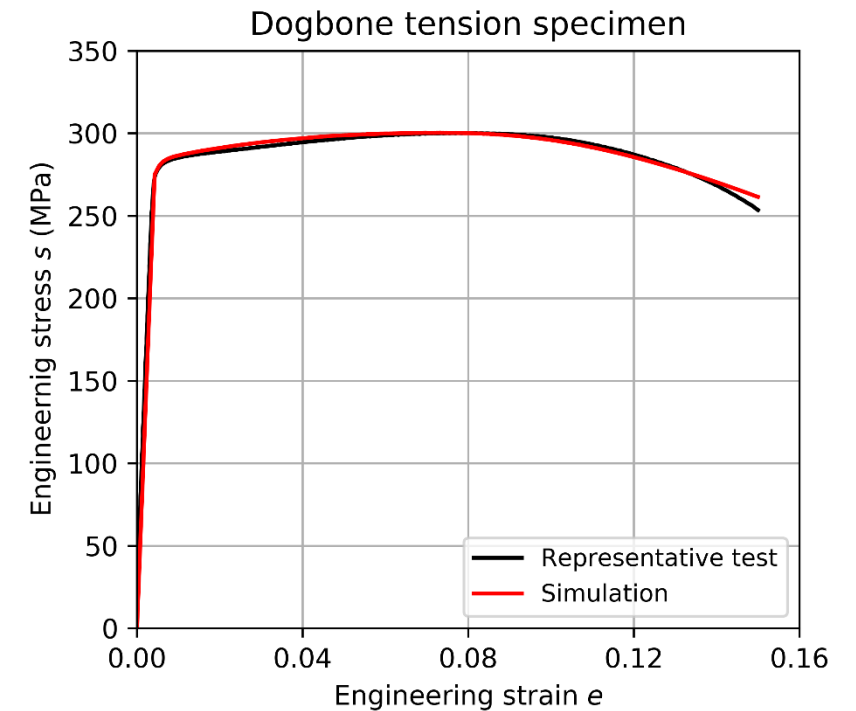
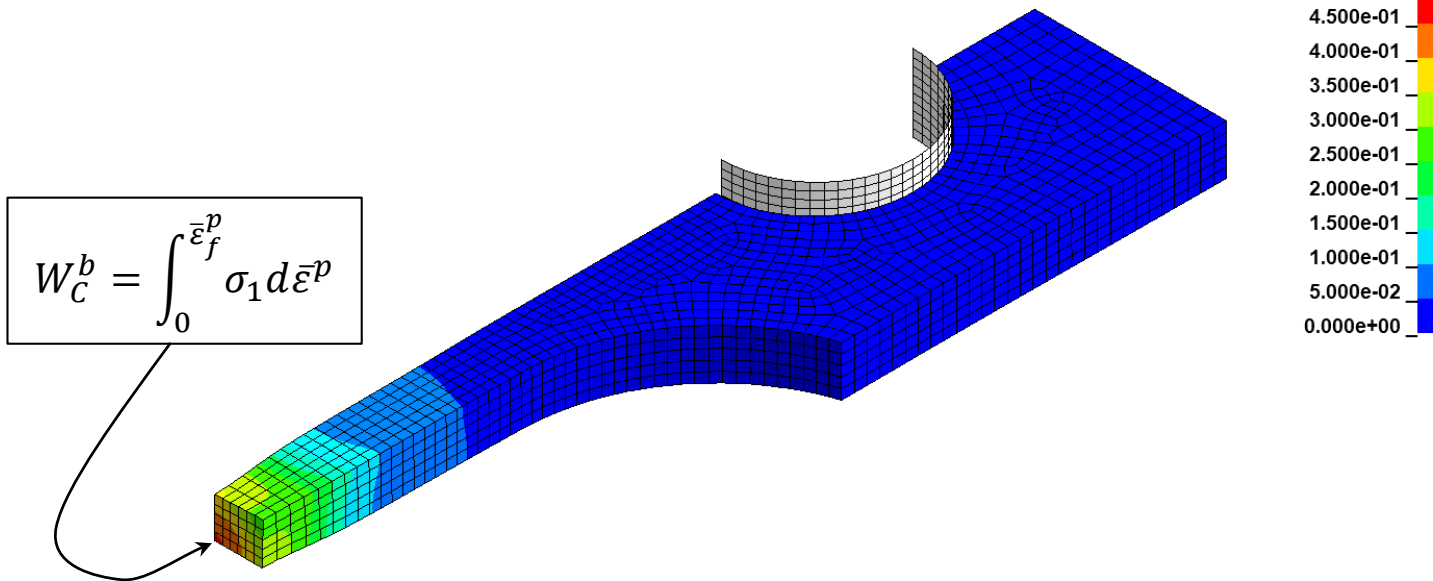
Solid element model

- Solid element model
 - XYZ symmetry
 - ELFORM = 1
 - Element size : ≈ 0.26 mm
 - Gauge length L_0 : 9.87 mm
- *MAT_033/
*MAT_BARLAT_ANISOTROPIC_PLASTICITY
 - A = B = C = D = E = F = G = H = 1.0
 - M = 8.0
 - Yield stress and hardening defined by LCID that was determined using LS-OPT
- Load the model with *DEFINE_CURVE_SMOOTH and a 20% ramp-up time



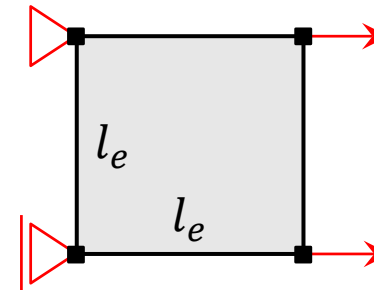
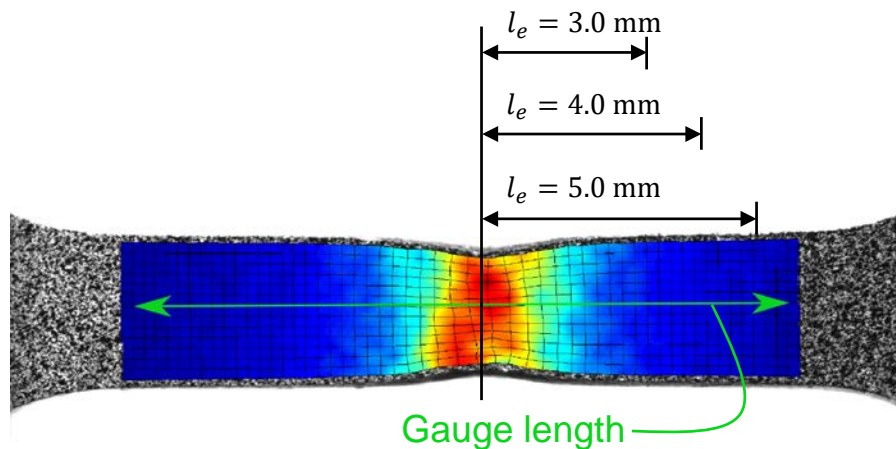
Solid element model

- Determine failure from the centermost element in a simulation of a tension test
- This is how we find W_C^b
- $W_C^b = 165.6 \text{ MPa}$



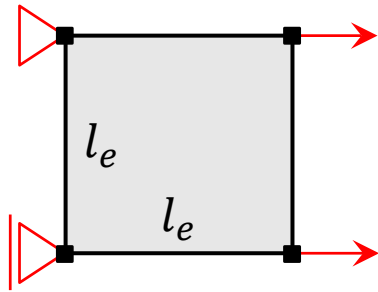
Single element models

- From the DIC analysis:
 - Find the elongations at fracture at various points away from the neck
 - **This definition is conservative**
- Stretch the single element models to the corresponding elongation at fracture and calculate the different W_C^m

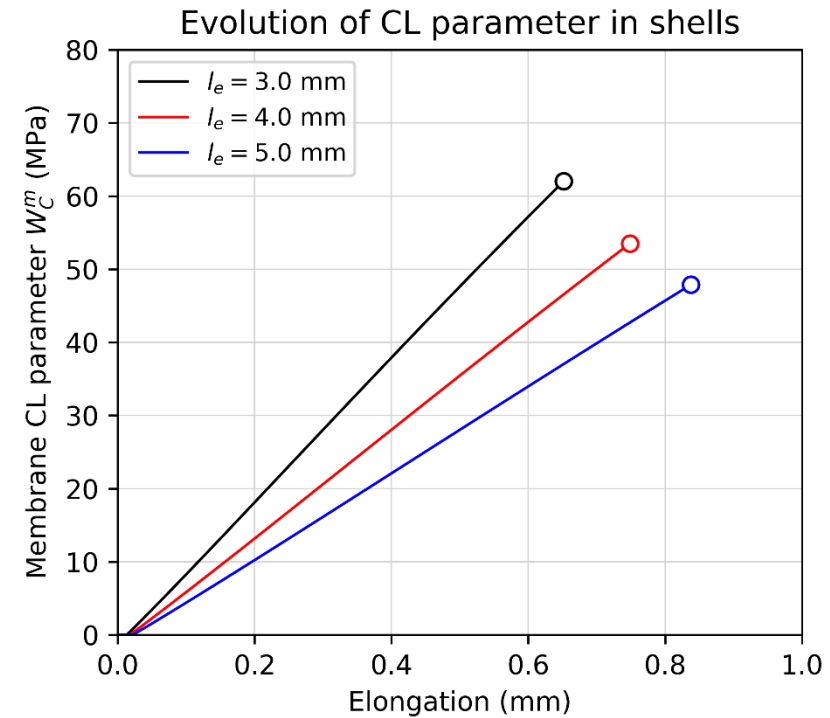


Shell element models

- Large elements → Low CL parameter (W_C^m)



l_e/t_e	W_C^m
1.17	62.1 MPa
1.56	53.4 MPa
1.95	48.0 MPa



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EFFECT OF REGULARIZATION AND FAILURE PARAMETERS

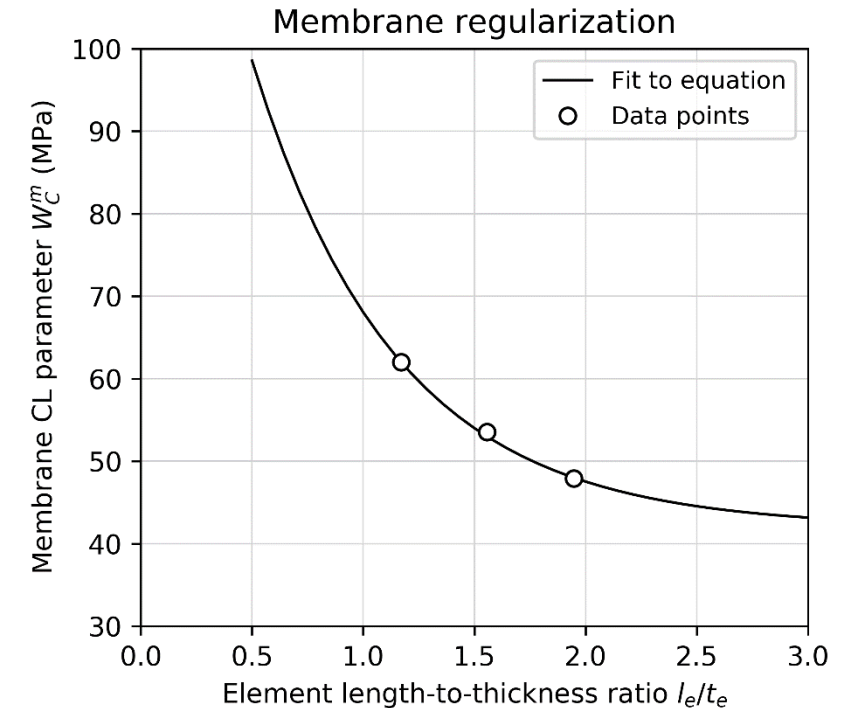
Element-size regularization

From 1-element models

$$W_C^m = W_C^l + (W_C^s - W_C^l) \exp\left(-c \left(\frac{l_e}{t_e} - 1\right)\right)$$

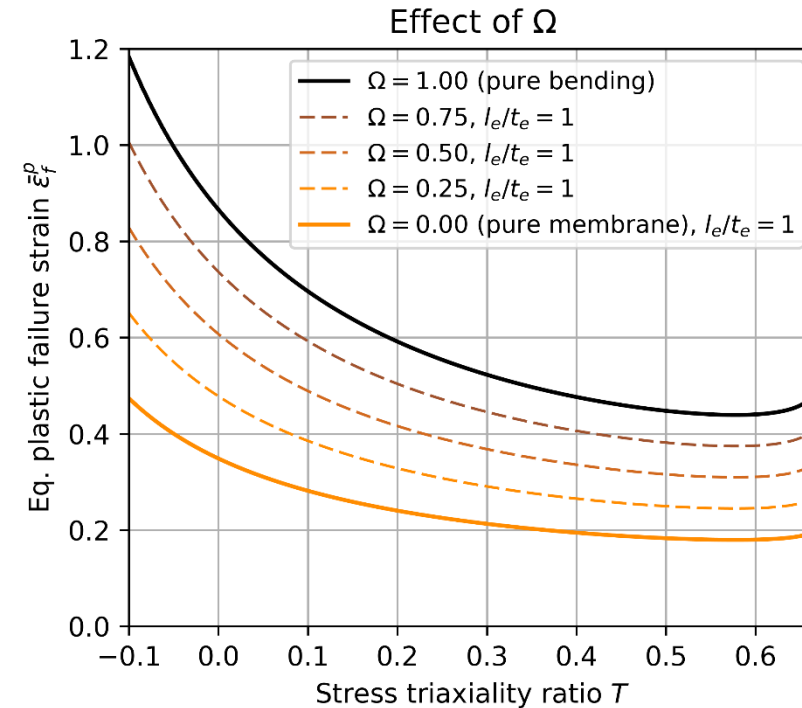
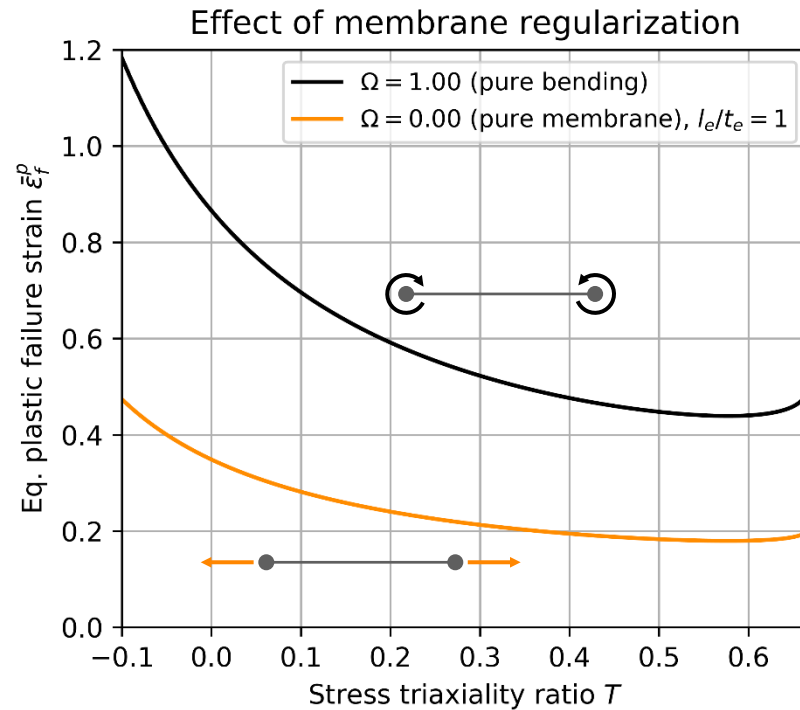
l_e/t_e	W_C^m
1.17	62.1 MPa
1.56	53.4 MPa
1.95	48.0 MPa

param	value
W_C^s	42.0 MPa
W_C^l	68.1 MPa
c	1.549



Membrane/bending regularization

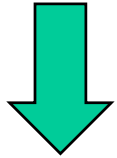
$$W_C = \Omega W_C^b + (1 - \Omega) W_C^m$$



Extended Cockcroft-Latham (ECL) failure

- ECL assumes that damage evolution is driven by plastic power amplified by a stress state dependent term

$$D = \frac{1}{W_C} \int_0^{\bar{\epsilon}_f^p} \left\langle \phi \frac{\sigma_I}{\bar{\sigma}} + (1 - \phi) \left(\frac{\sigma_I - \sigma_{III}}{\bar{\sigma}} \right) \right\rangle^{\gamma} \bar{\sigma} d\bar{\epsilon}^p$$



Invariant expression. Used to plot the failure surface

$$W_C = \int_0^{\bar{\epsilon}_f^p} \left\langle \frac{\phi(3T\sqrt{3+L^2} - 3 - L)}{3\sqrt{3+L^2}} \right\rangle^{\gamma} \bar{\sigma} d\bar{\epsilon}^p$$

$$D = \frac{1}{W_C} \int_0^{\bar{\epsilon}_f^p} \langle \sigma_I \rangle d\bar{\epsilon}^p \quad \text{Cockcroft-Latham criterion } (\phi = \gamma = 1)$$

$$D = \frac{1}{W_C} \int_0^{\bar{\epsilon}_f^p} \bar{\sigma} d\bar{\epsilon}^p \quad \text{Freudenthal criterion } (\gamma = 0)$$

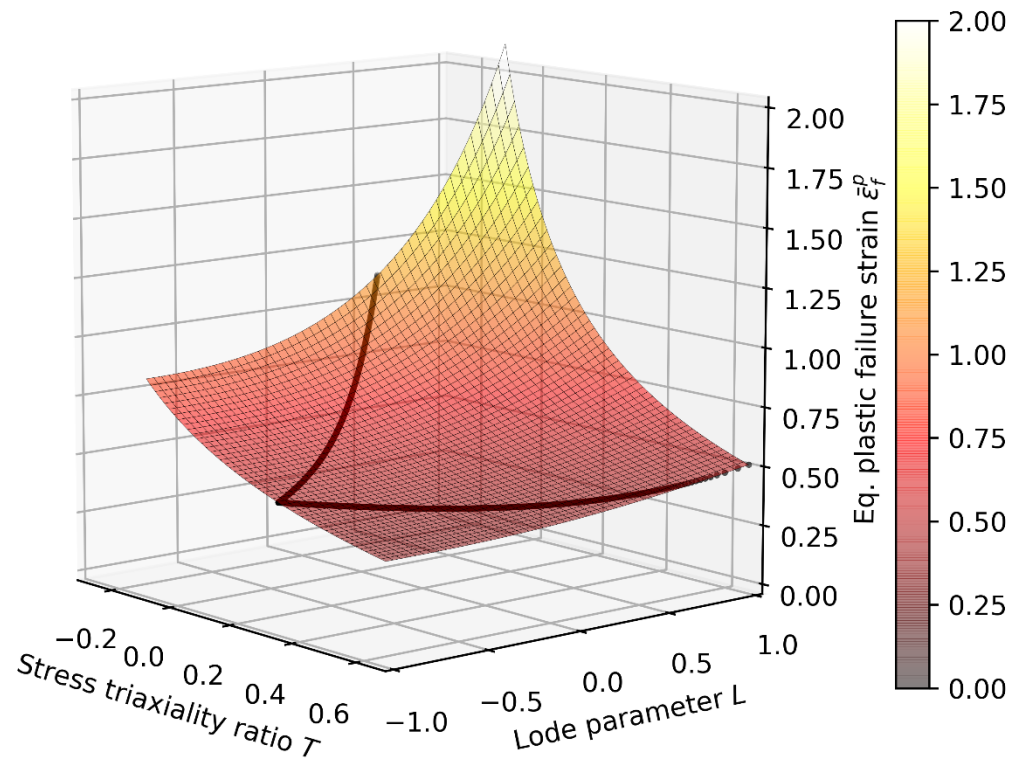
$$D = \frac{1}{W_C} \int_0^{\bar{\epsilon}_f^p} (\sigma_I - \sigma_{III}) d\bar{\epsilon}^p \quad \text{Integral based Tresca criterion } (\phi = 0, \gamma = 1)$$

Gruben G, Hopperstad OS, Børvik T. *Evaluation of uncoupled ductile fracture criteria for the dual-phase steel Docol 600DL*. International Journal of Mechanical Sciences 2012; 62: 133-146.

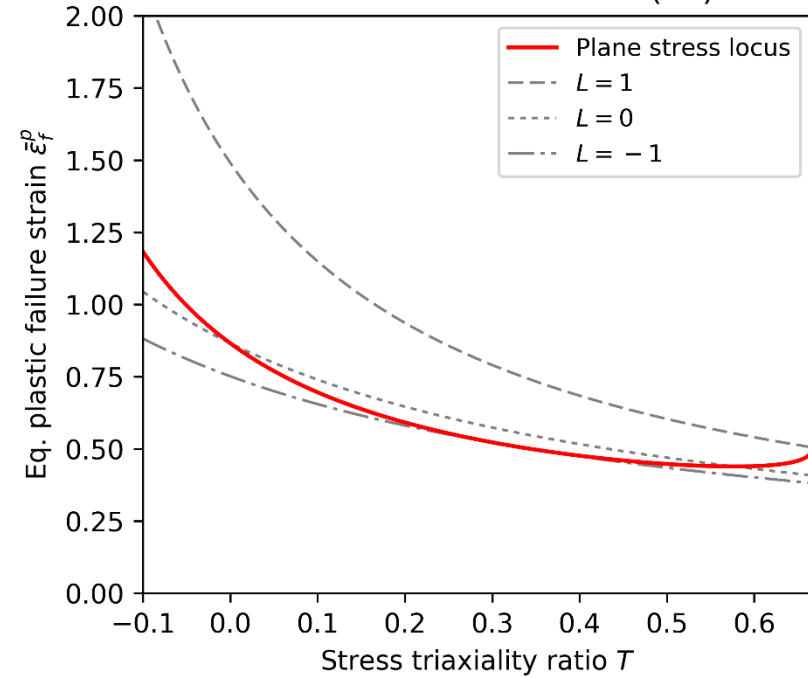
Failure locus for CL failure

$$W_C = 165.6 \text{ MPa}, \gamma = 1, \varphi = 1$$

3D failure locus (CL)

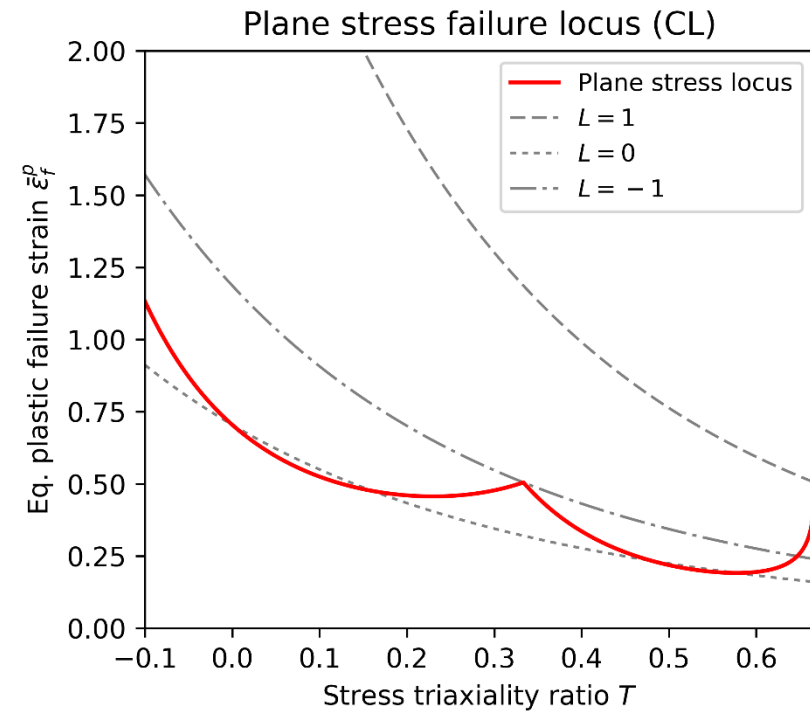
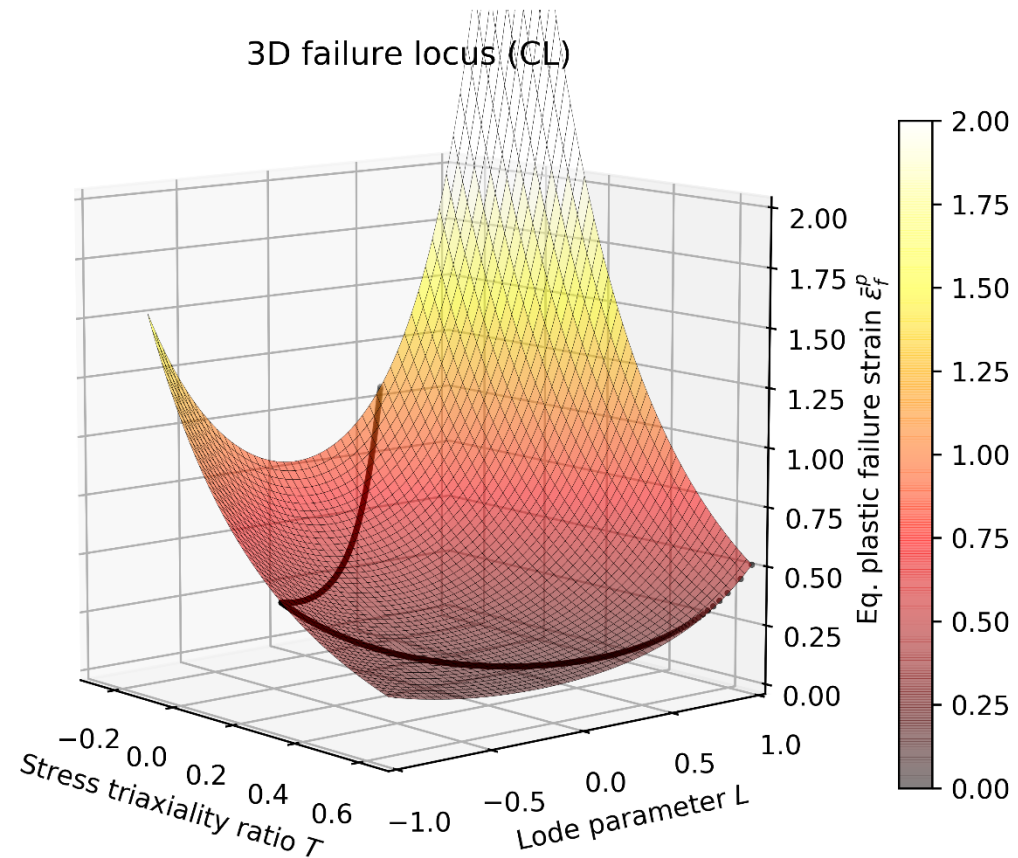


Plane stress failure locus (CL)



Failure locus for ECL failure

$$W_C = 165.6 \text{ MPa}, \gamma = 0.35, \varphi = 7.0$$



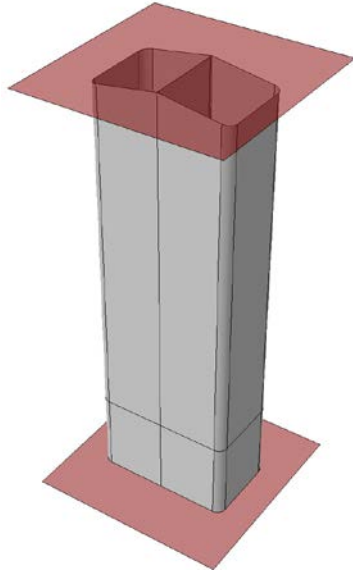
*MAT_258 material card

TITLE							
Aluminum 6005-T6							
Yielding	<u>MID</u>	<u>RO</u>	<u>E</u>	<u>PR</u>	<u>SIGY</u>	<u>A</u>	<u>KSI</u>
	1	2.7e-09	70.0e+03	0.3	275.68	8.0	0.0
Hardening	<u>THETA1</u>	<u>Q1</u>	<u>THETA2</u>	<u>Q2</u>	<u>THETA3</u>	<u>Q3</u>	
	7095.1	8.61	702.3	48.47	166.2	12.16	
Rate sensitivity	<u>CS</u>	<u>PDOTS</u>					
	0.0	0.0					
Failure	<u>DCRIT</u>	<u>WCB</u>	<u>WCL</u>	<u>WCS</u>	<u>CC</u>	<u>PHI</u>	<u>GAMMA</u>
	1.0	165.6	42.0	68.1	1.549	1.0	1.0
							THICKNESS
							2.57

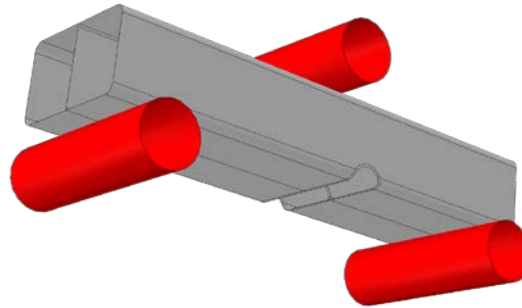
5

VALIDATION

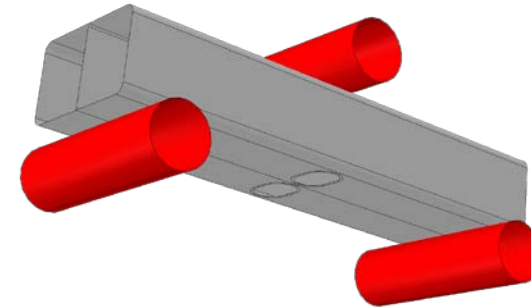
Validation tests



Quasi-static crushing



**3-point-bending
w/notch**

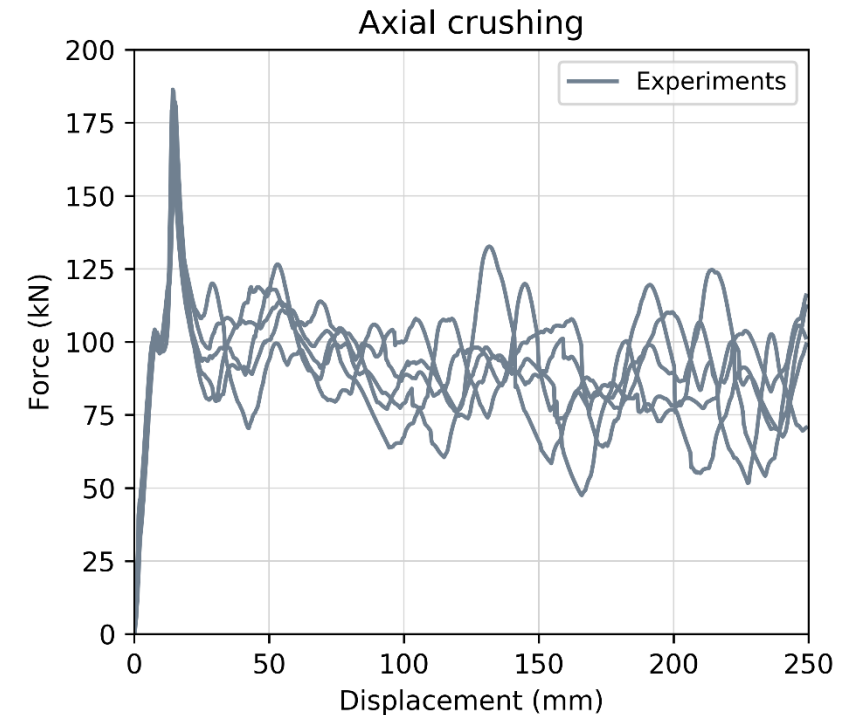


**3-point-bending
w/holes**

Costas M, Morin D, Hopperstad OS, Børvik T, Langseth M. *A through-thickness damage regularization scheme for shell elements subjected to severe bending and membrane deformations*. Journal of the Mechanics and Physics of Solids. In press.

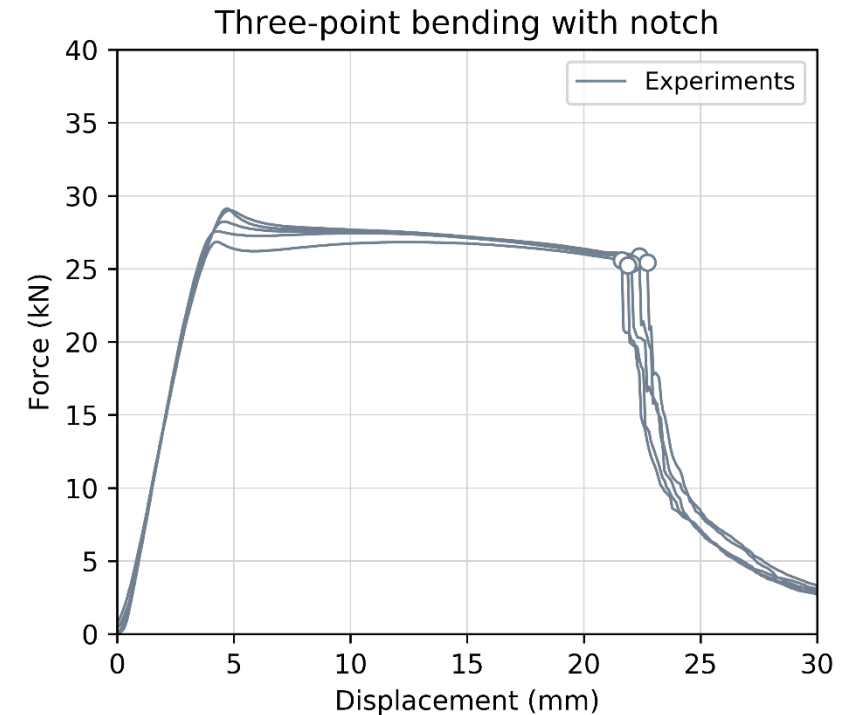
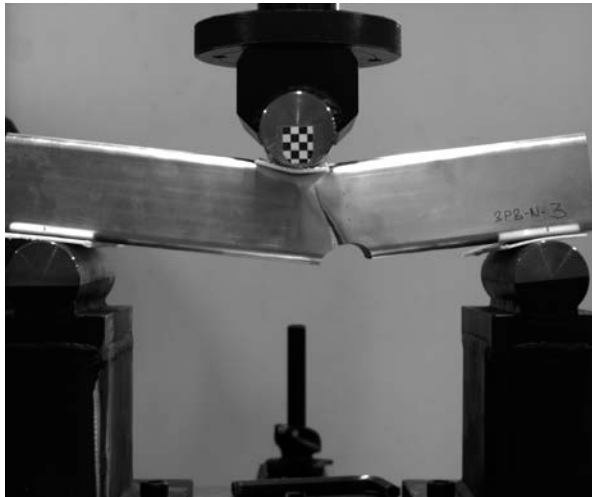
Axial crushing tests

- Five successful repetitions
- Cracks at the corners and along both ends of the middle wall
- Collapse mode transitions from asymmetric to symmetric



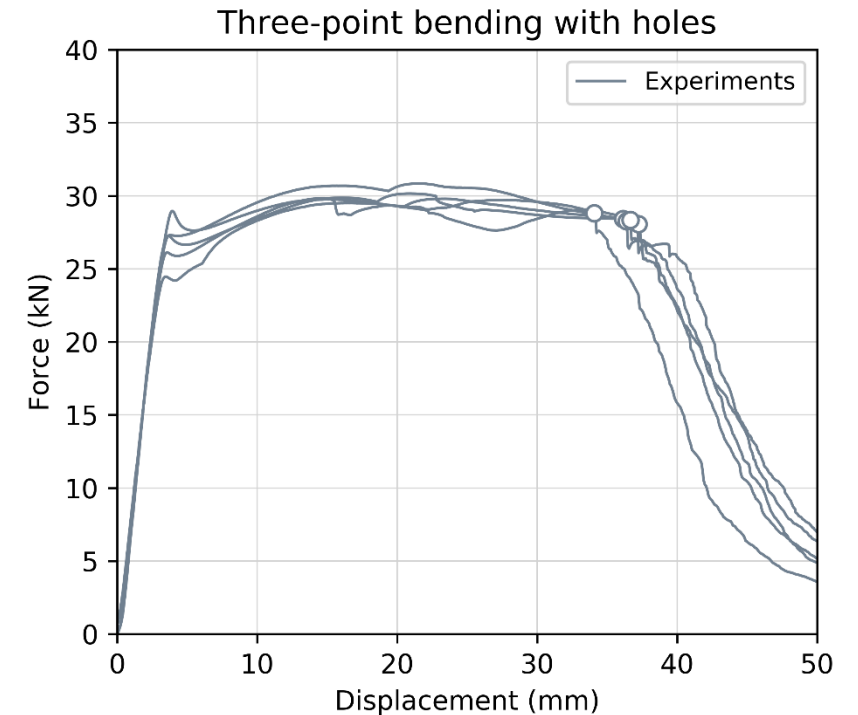
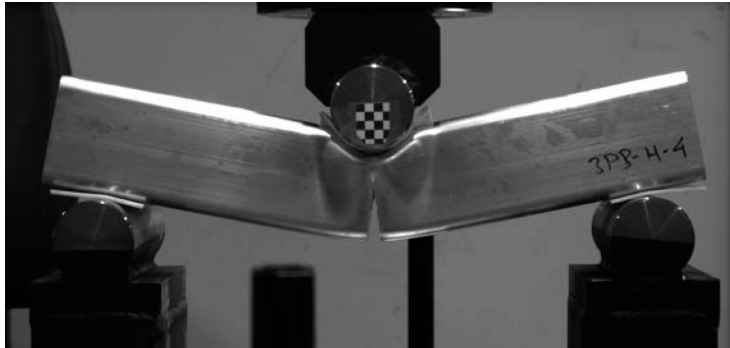
3-point-bending tests with notch

- Failure takes place after approx. 23 mm displacement
- Inclined crack-growth from the notch toward the punch



3-point-bending tests with holes

- Failure takes place after approx. 35 mm displacement
- Straight crack-growth from the holes toward the punch



Modeling details

*CONTACT_AUTOMATIC_SINGLE_SURFACE

- Use LSTC recommended values
 - ([SOFT](#) = 1, [VDC](#) = 20)

Element type and hourglass formulation

- Use default [ELFORM](#)=2 and LSTC recommended values [IHQ](#) = 4, [QM](#) = 0.03
- Compare to [ELFORM](#)=16 and [IHQ](#) = 8, [QM](#) = 0.03

Updating of shell normals

- *Default:*
 - Recompute fiber directions at every cycle, [IRNXX](#) = -1
- *Alternative:*
 - Unique nodal fibers are incrementally updated based on the nodal rotation at the location of the fiber, [IRNXX](#) = -2

Modeling details

Three simulation setups presented in the following:

“ELFORM = 2”:

Simulations with ELFORM = 2 and IRNXX = -1

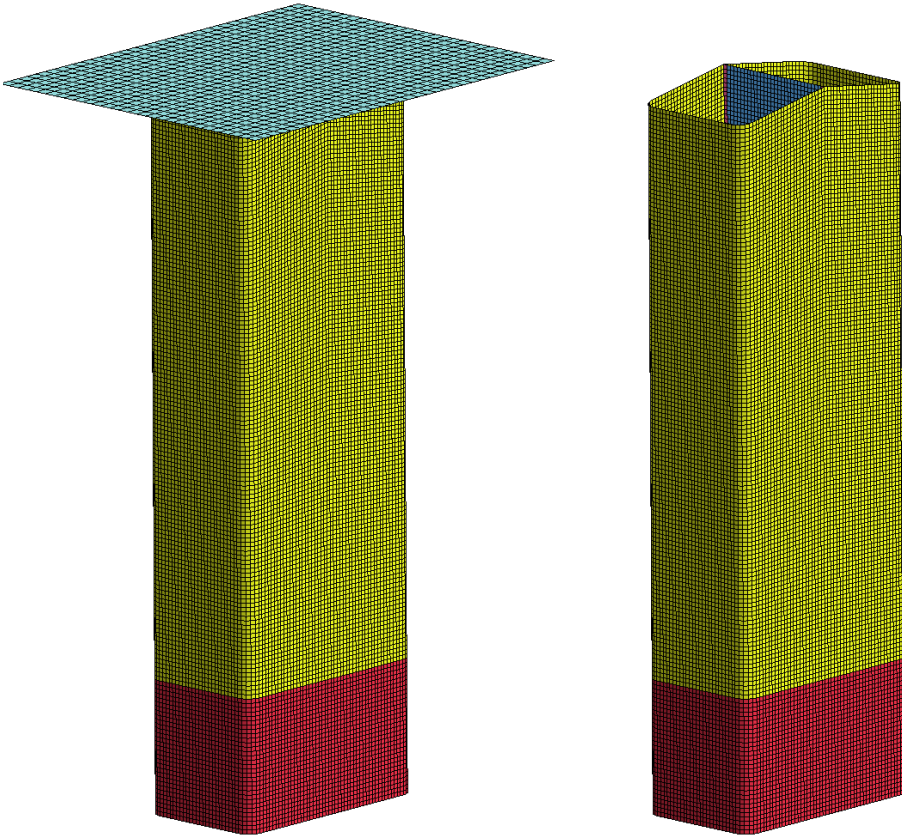
“ELFORM = 16”:

Simulations with ELFORM = 16 and IRNXX = -1

“IRNXX = -2”:

Simulations with ELFORM = 2 and IRNXX = -2

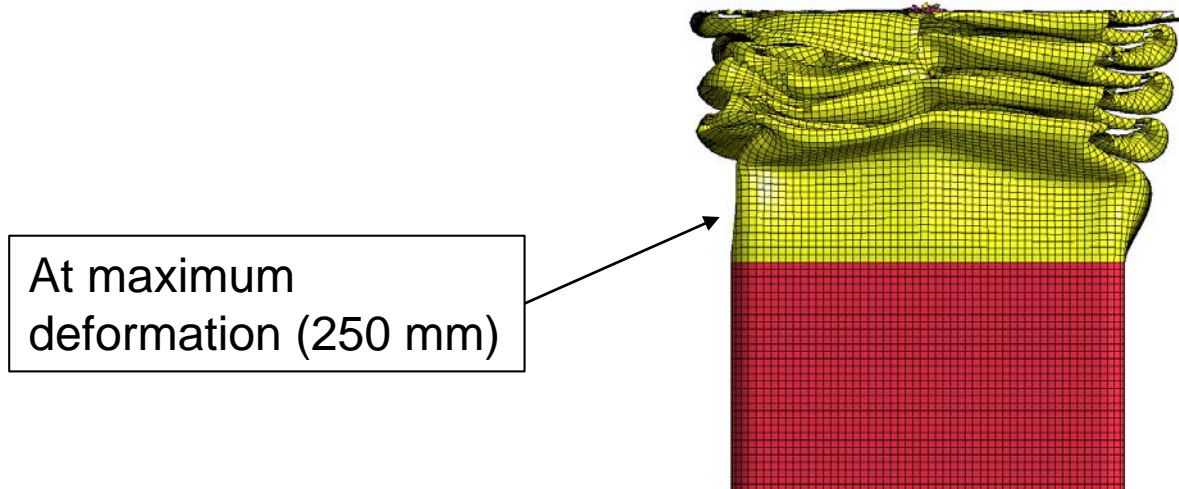
Axial crushing



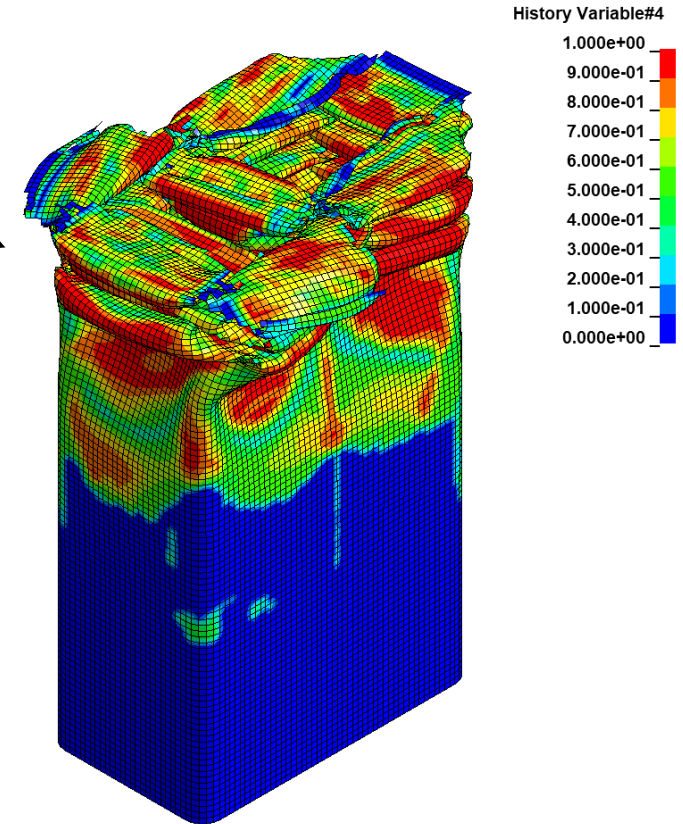
- Elem. size ≈ 2.7 mm
- 350 mm free length (incl. 9° trigger)
- 80 mm clamping length
- 250 mm crushing distance
- Time scaling factor of 25 000
 - Negligible kinetic energy. Lower factor did not affect the results
- Smooth ramping of the load
- Friction coefficient $\underline{FS} = \underline{FD} = 0.2$
- Account for thickness change ($\underline{ISTUPD} = 1$)
- Rigid punch (teal) and bottom part (red) of profile

Axial crushing

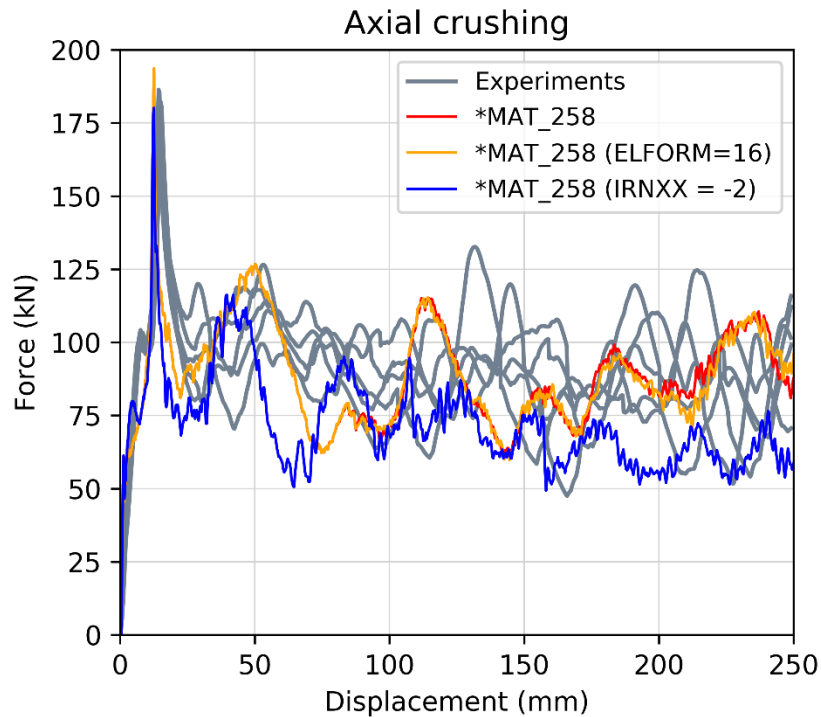
- Pictures of deformed shapes after significant deformation
- Predict more fracture in the simulation than in the test
- History variable #4 = Ω
- Red = pure bending
- Blue = pure membrane



At intermediate deformation (194 mm)



Axial crushing

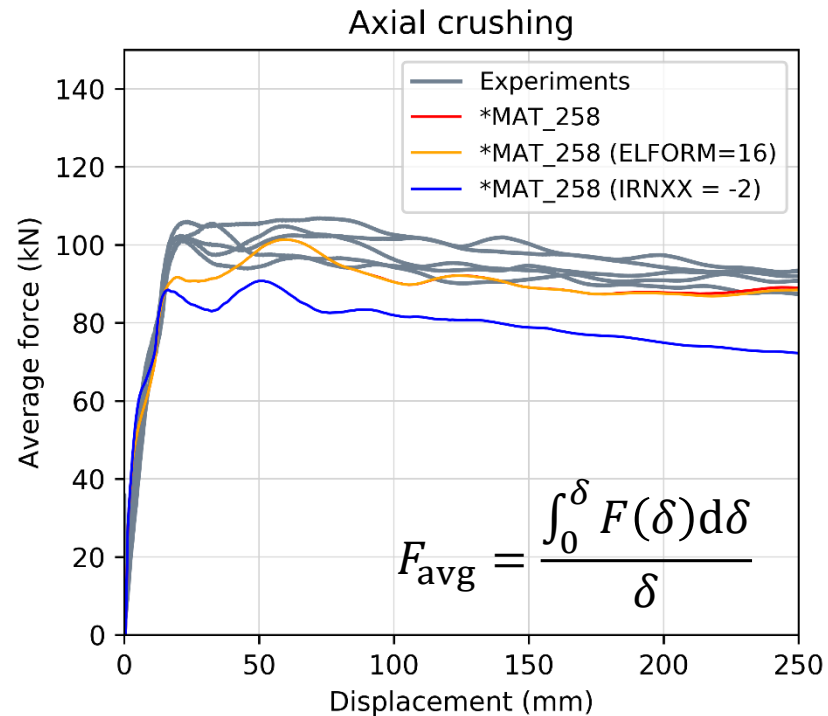


Force-displacement curves:

ELFORM=2 and ELFORM=16 give almost identical results.

IRNXX=2 generates a completely different collapse mode

Axial crushing



Average force-displacement:

ELFORM = 2 and ELFORM = 16:

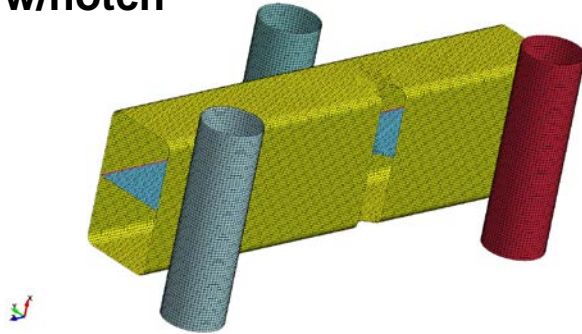
Mean force within experimental scatter starting at a displacement of 40 mm

IRNXX = -2:

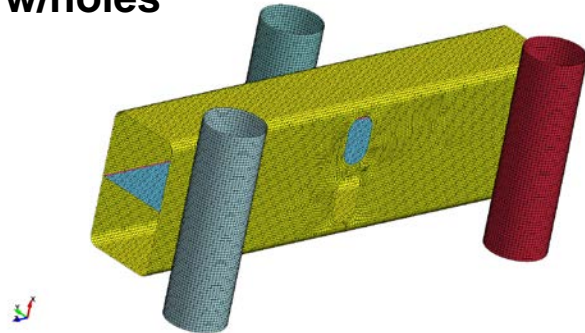
Mean much too low due to a peeling collapse mechanism instead of a buckling mechanism

3-point-bending

w/notch



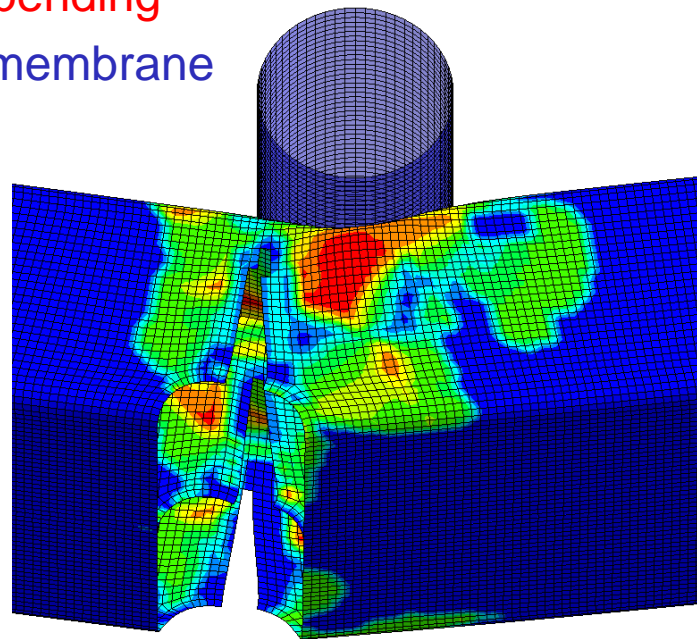
w/holes



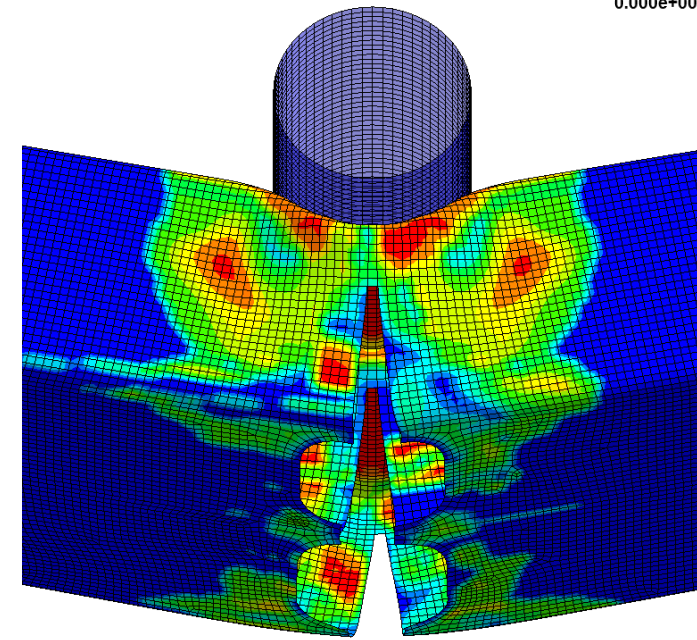
- Elem. size ≈ 2.7 mm
- 400 mm between the supports
- Punch is centered
- Time scaling factor of 3000
 - Negligible kinetic energy. Lower factor did not change the results
- Smooth ramping of the load
- Friction coefficient $\underline{FS} = \underline{FD} = 0.05$
- Account for thickness change ($\underline{ISTUPD} = 1$)
- Rigid punch and supports

3-point-bending

- Pictures of deformed shapes after significant deformation
- Crack growth similar, but constrained by element size
- History variable #4 = Ω
 - Red = pure bending
 - Blue = pure membrane

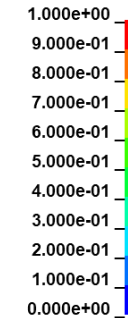


w/notch

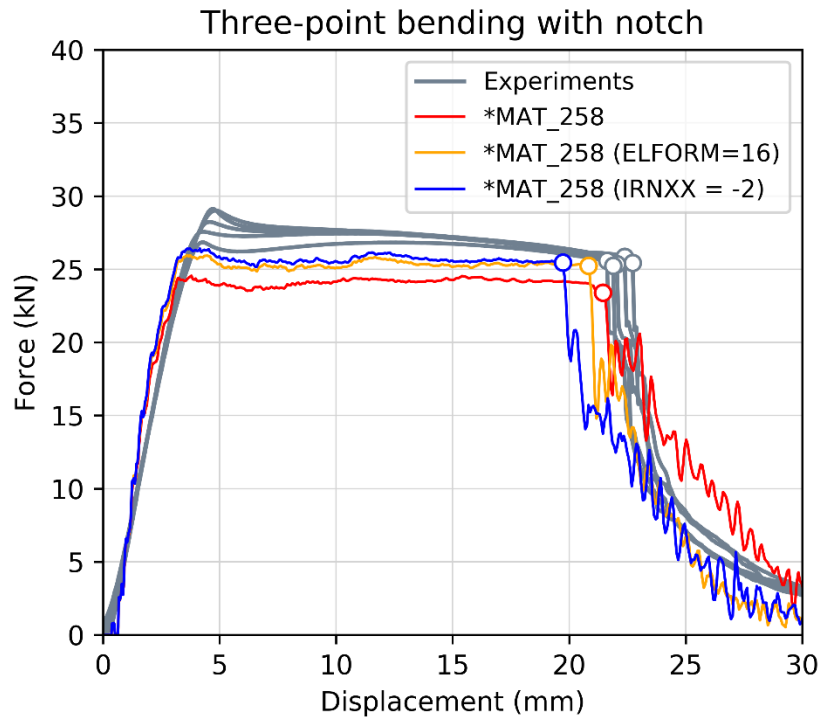


w/holes

History Variable#4



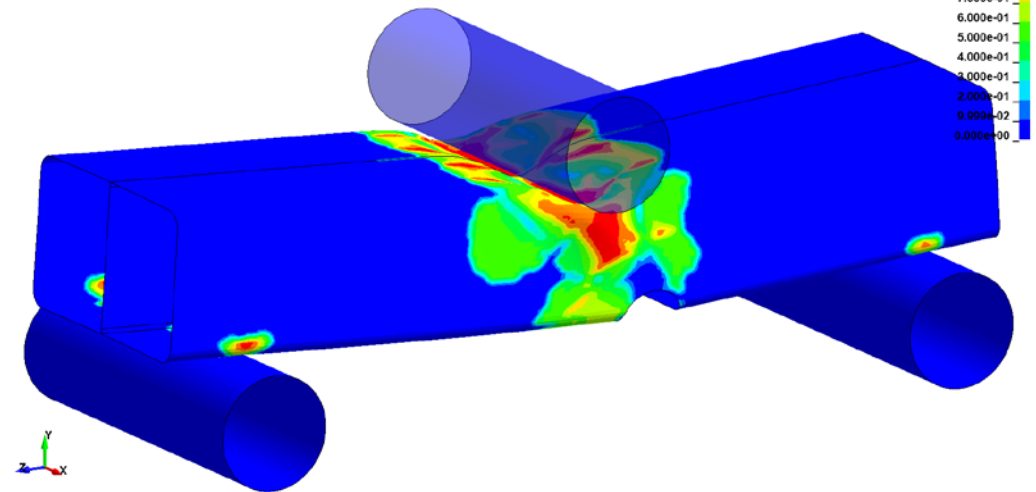
3-point-bending with notch



ELFORM = 2: Low force and decent point of failure.

ELFORM = 16 and IRNXX = -2: Higher force which induces earlier failure due to increased growth of D .

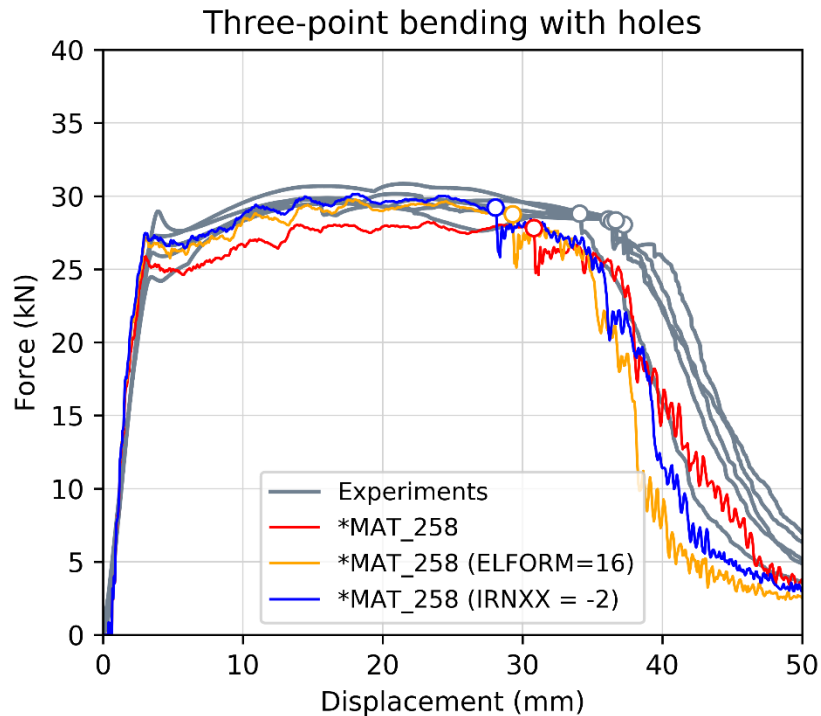
LS-DYNA keyword deck by LS-PrePost
Time = 0.046
Contours of History Variable#4
reference shell surface
min=0, at elem# 1
max=0.999949, at elem# 15804



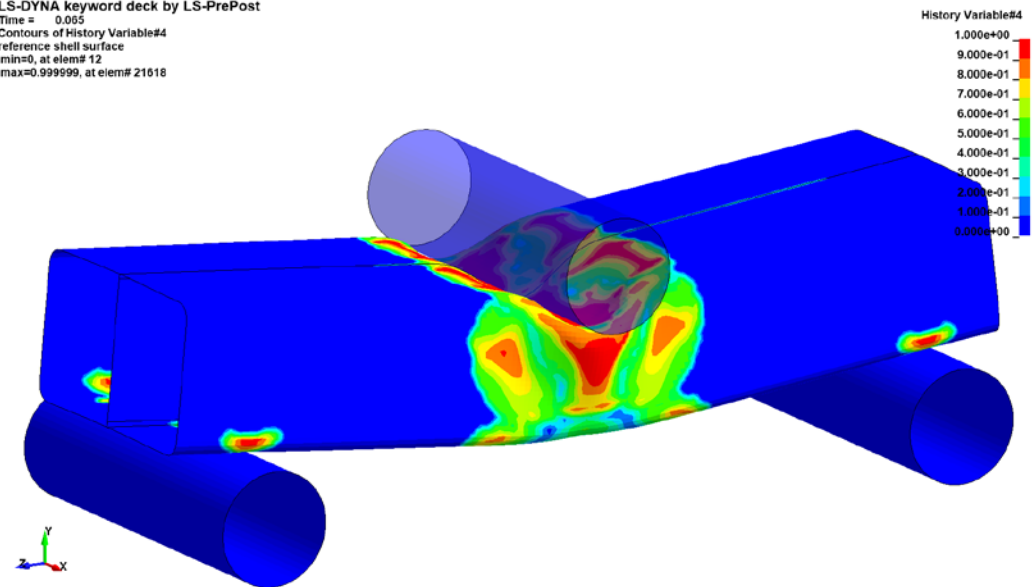
3-point-bending with holes

ELFORM = 2: Low force and early failure.

ELFORM = 16 and IRNXX = -2: Higher force which induces earlier failure due to faster growth of D .

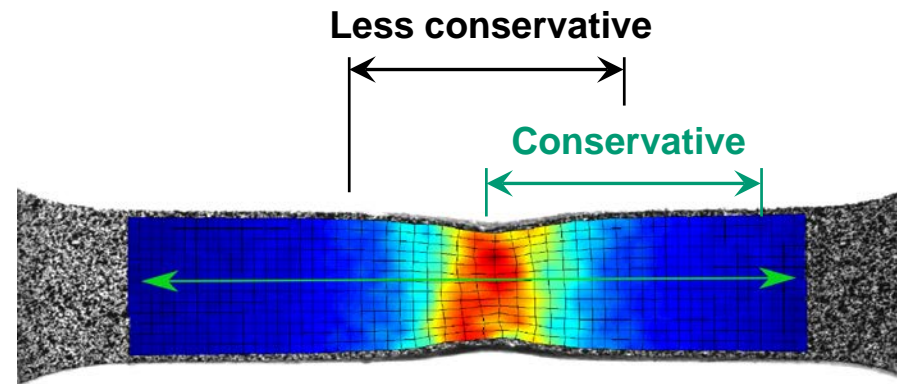


LS-DYNA keyword deck by LS-PrePost
Time = 0.055
Contours of History Variable#4
reference shell surface
min=0, at elem# 12
max=0.999999, at elem# 21618



Summary – simulation results

- The mean force is **strongly** dependent upon the element formulation
- Failure initiation is conservative, which was the **intention** of the model
- Superimposing the shell elements used for calibration over the entire notch, instead of half the notch, will give less conservative results

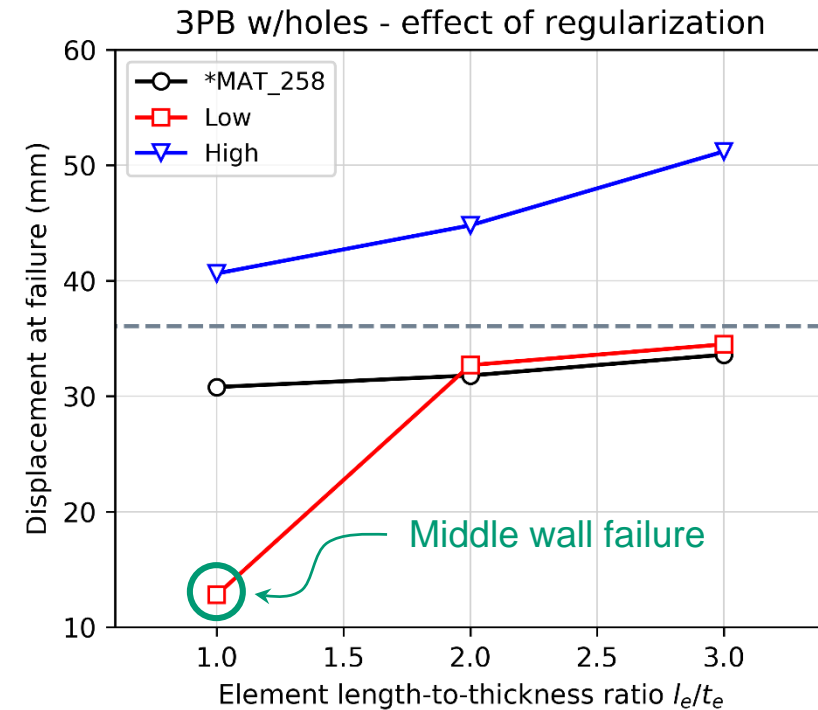
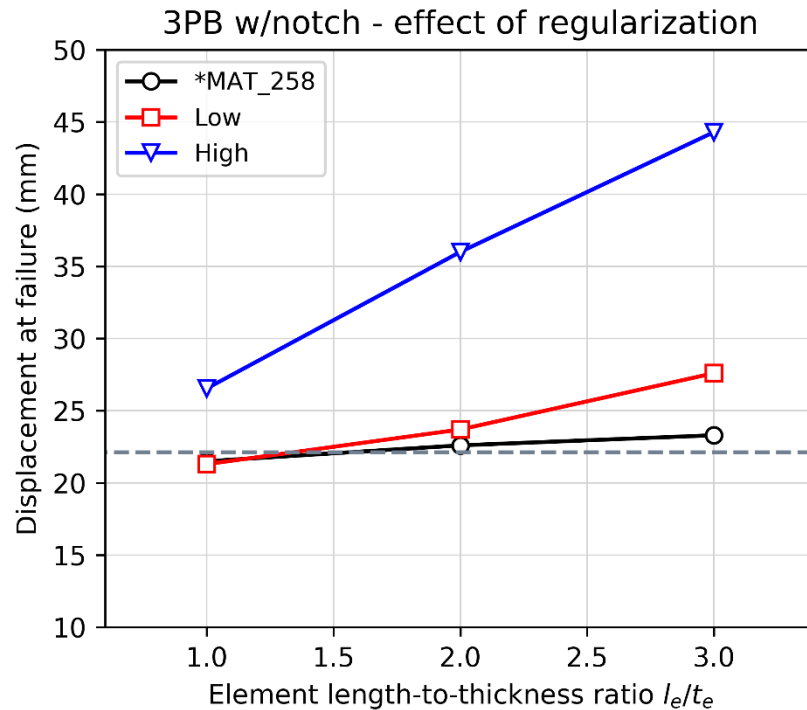


6

COARSER MESHES

Coarser meshes

- Simulations with $l_e/t_e = 1, 2$ and 3 for three-point bending with holes and notch
- **Low** : Used $W_C = W_C^m$ for $l_e/t_e = 1$ (**68.1 MPa**)
- **High**: Used $W_C = W_C^b$ from the solid element model (**165.6 MPa**)



8

SUMMARY

Model summary

- The flexible hardening rule can represent a variety of hardening curves
- Hosford-Hershey yield function can represent von Mises, Tresca, and everything in between
- The ECL failure criterion offers great flexibility with few parameters
- Viscoplasticity is implemented
- Simple calibration
 - In this presentation is was calibrated with only one tension test
 - For more control of the failure locus, more tests can be used

References

- Costas M, Morin D, Hopperstad OS, Børvik T, Langseth M. *A through-thickness damage regularization scheme for shell elements subjected to severe bending and membrane deformations*. Journal of the Mechanics and Physics of Solids. In press.
- Gruben G, Hopperstad OS, Børvik T. *Evaluation of uncoupled ductile fracture criteria for the dual-phase steel Docol 600DL*. International Journal of Mechanical Sciences 2012; 62: 133-146.