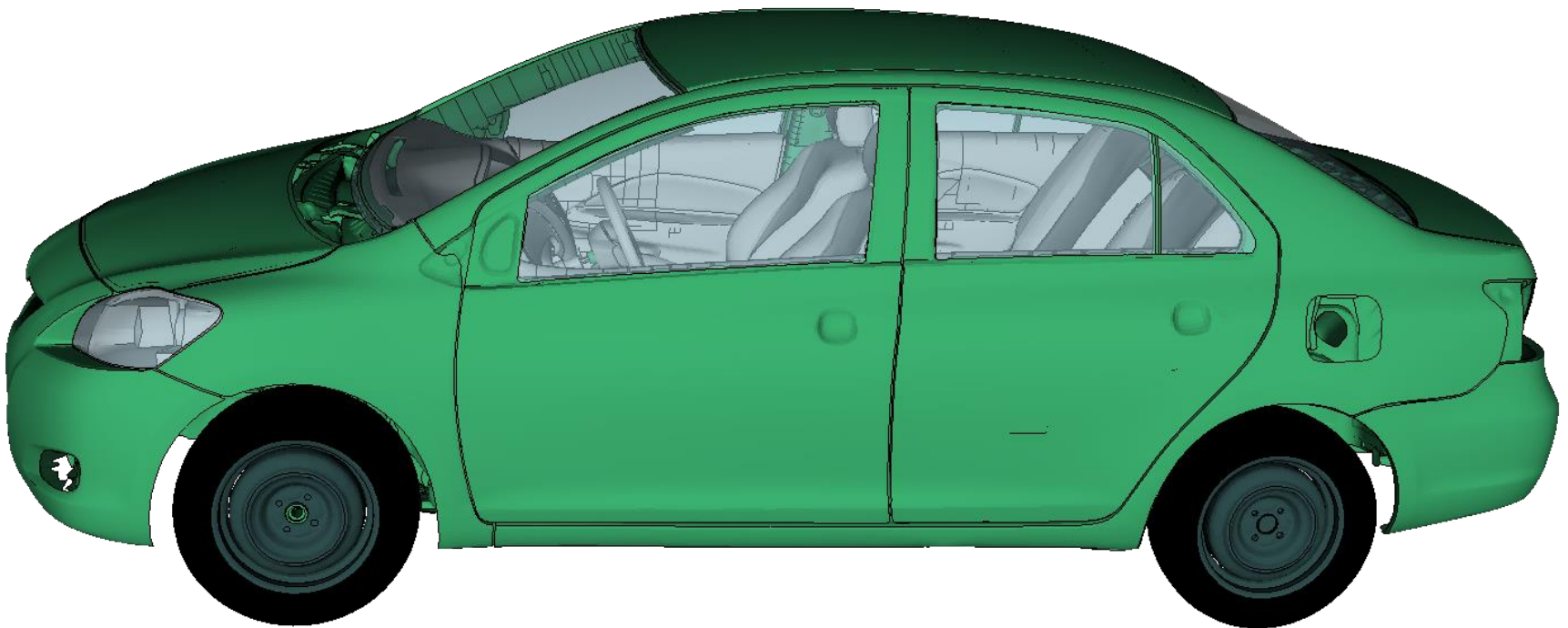


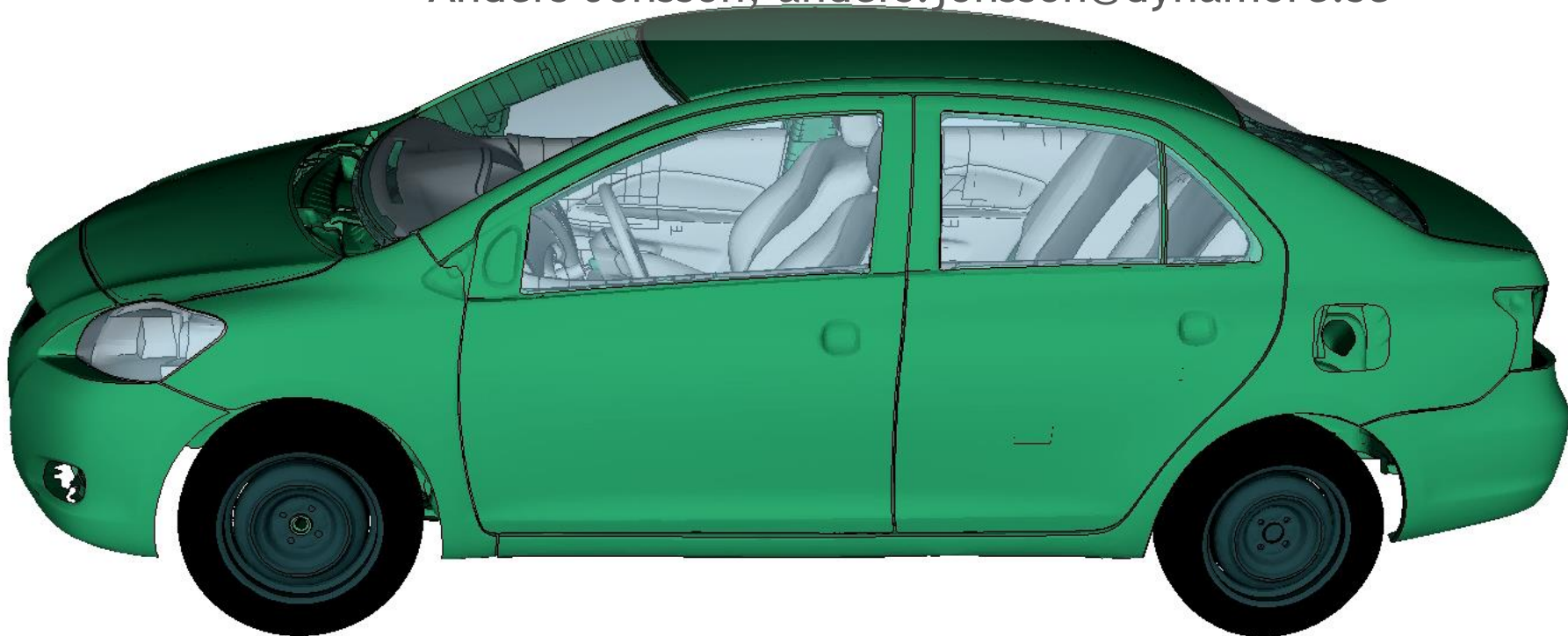
From explicit to implicit - re-using crash models for static load cases with a minimal effort

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From explicit to implicit - re-using crash models for static load cases with a minimal effort

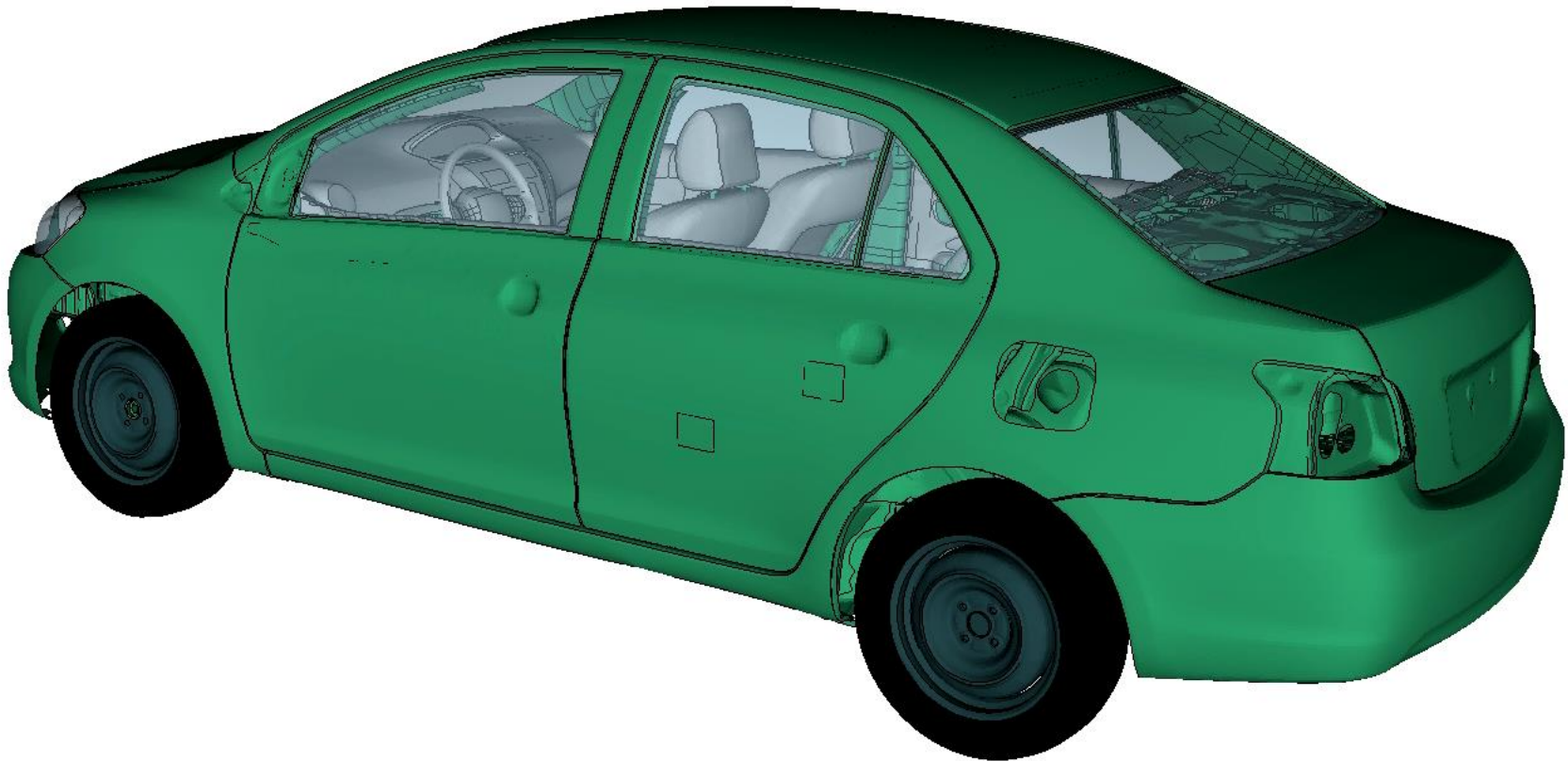
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LS-DYNA implicit - Brief summary

- LS-DYNA Implicit works well
 - Also for large scale models, single surface contacts, material failure etc.
- It's possible to convert a large-scale explicit crash model to run in implicit
- LSTC and Dynamore are continuously working on making implicit more useful and easy-to-use
 - LS-PrePost GUI for simplified set-up of implicit analyses
 - LS-DYNA Bundle, including tutorials and guidelines
 - Guideline for implicit analyses, including examples

LS-DYNA implicit - Opening for new possibilities



LS-DYNA implicit - Opening for new possibilities

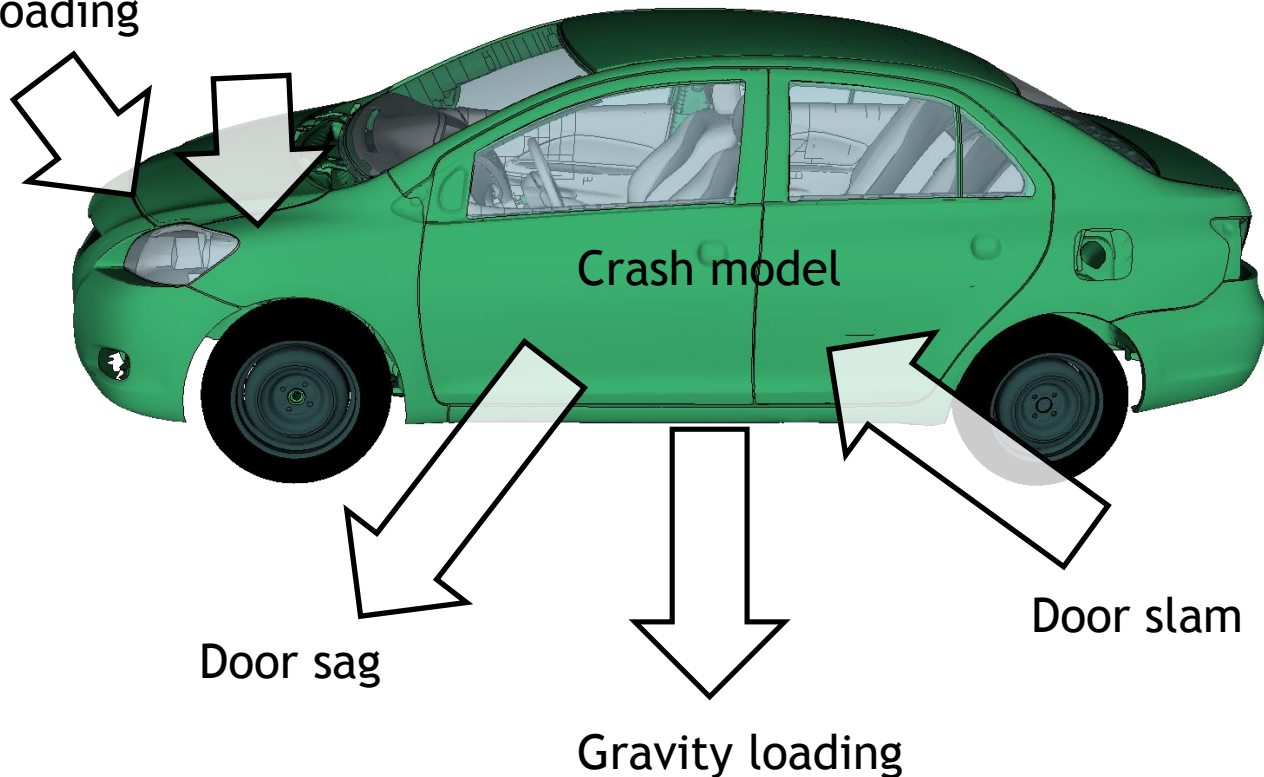
- This presentation will briefly demonstrate how a crash model can be re-used for static load cases using the implicit capabilities of LS-DYNA
- The objective was minimal effort for model conversion from explicit. No parts were removed from the car model.
 - (Local) re-meshing may be required for some load cases, for example if highly resolved stresses would be desired in critical areas

Examples in this presentation based on a [public](#) FE-model of 2010 [Toyota Yaris](#) developed by the Center for Collision Safety and Analysis (CCSA) at the George Mason University (GMU) under a contract with the Federal Highway Administration (FHWA)

LS-DYNA implicit - Opening for new possibilities

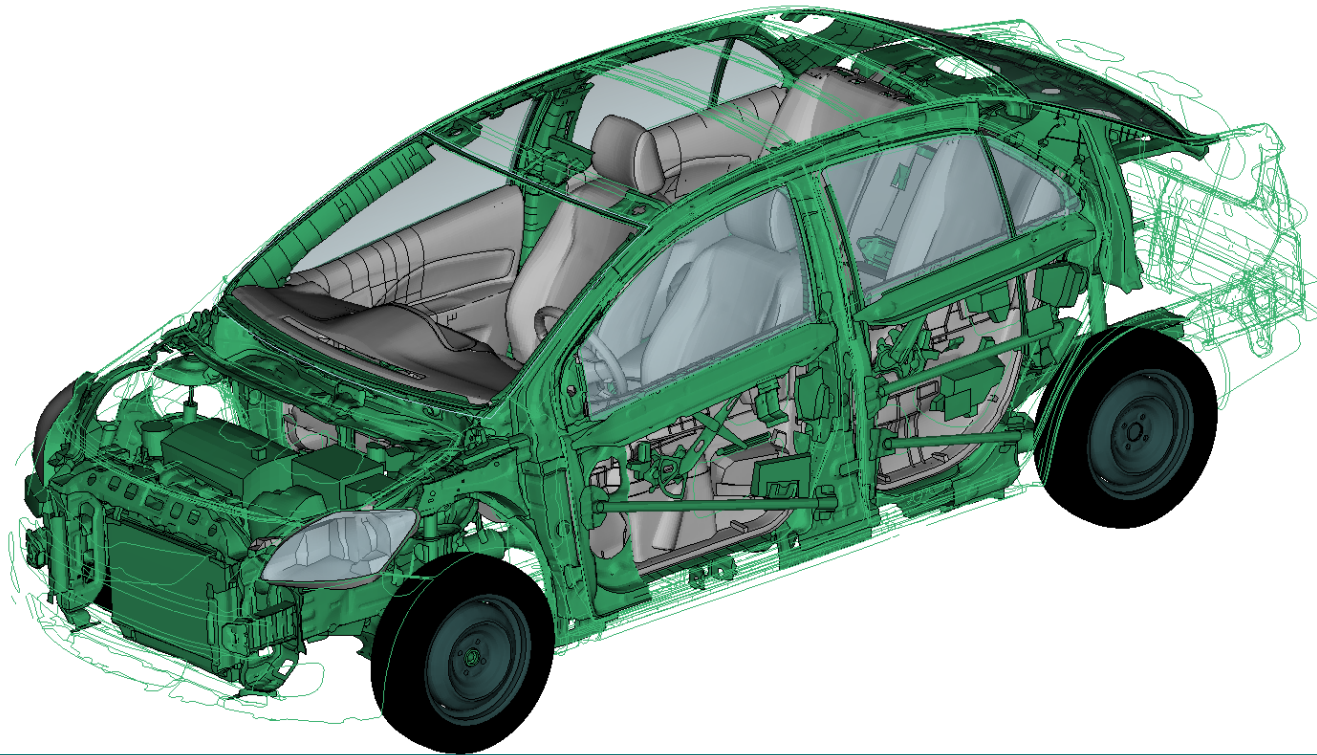
- Start out with a model for explicit crash analysis
- Create a model that works in implicit by Minimal modifications
 - Many modifications could also be included in the crash model

Fender loading



Model data

- Examples in this presentation based on a [public](#) FE-model of 2010 [Toyota Yaris](#) developed by the Center for Collision Safety and Analysis (CCSA) at the George Mason University (GMU) under a contract with the Federal Highway Administration (FHWA)
 - Note: Complete model incl. engine and interior trim was used



Model data

- Examples in this presentation based on a [public](#) FE-model of 2010 [Toyota Yaris](#) developed by the Center for Collision Safety and Analysis (CCSA) at the George Mason University (GMU) under a contract with the Federal Highway Administration (FHWA)
 - Note: Complete model incl. engine and interior trim was used
- The same source was also used for similar examples on www.dynaexamples.com/implicit
- Model size is approximately 1.5 M elements. Car weight is 1100 kg, $L \approx 4.3$ m, $w \approx 1.7$ m, $h \approx 1.5$ m.
- The different examples are run in mpp-LS-DYNA R9.1 or R10.0, using double precision on 16 - 24 cores.

LS-DYNA implicit set up

- The implicit load cases were set-up using the Guideline for implicit analyses.
 - **NEW Revision out now!** Download it from www.dynasupport.com/howtos/implicit/some-guidelines-for-implicit-analyses-using-ls-dyna
- It provides
 - recommended settings for different analysis types,
 - recommended element formulations, materials,
 - some small examples
 - and a trouble-shooting guide for convergence problems
- Also the LSTC Bundle contains some material regarding implicit analyses

Explicit to implicit - Modifications

- Unconnected parts or assemblies will cause rigid body modes, which may prevent convergence in implicit statics
- Check model connectivity!
 - Perform an eigenvalue analysis. Just add `*CONTROL_IMPLICIT_EIGENVALUE`
 - Use Check > Connectivity > Detect unconnected assemblies in ANSA
 - Check tied contacts. Setting `IPBACK = 1` on `*CONTACT_TIED_...` may be a quick fix for avoiding loose spot-welds
- Connectivity causing hinges or mechanisms
 - For example beam -> solid using common nodes
 - A CNRB connecting to one node of a solid will also cause a spherical joint
 - Joints
- Check for unsupported features
 - User defined material models
 - For example, `*MAT_TABULATED_JOHNSON_COOK` is not supported in implicit
- General model QA
 - Check mesh quality, initial penetrations etc.

Specific modifications: Yaris

- Removed “dummies” from crash model
- The tire airbags were a separated and switched to `*AIRBAG_LOAD_CURVE`
- The suspension was modified to `*ELEMENT_DISCRETE_LCO`
- Spherical joints in the steering were replaced by CNRBs. Some “spinning beams” in the front suspension were constrained
- Added three springs between exhaust system and BiW to reduce rotations
- Added CRNBs between radiator and side tube to eliminate hinge

*** Warning 60301 (IMP+301)

Using `*CONSTRAINED_SPOTWELD` with nodes without rotational dofs.

- Added CNRBs between windows and doors (to compensate for missing rubber seals)
- For the door-related load cases, the door hinges were aligned and some CNRBs between the BiW and the door were removed

Specific modifications for implicit

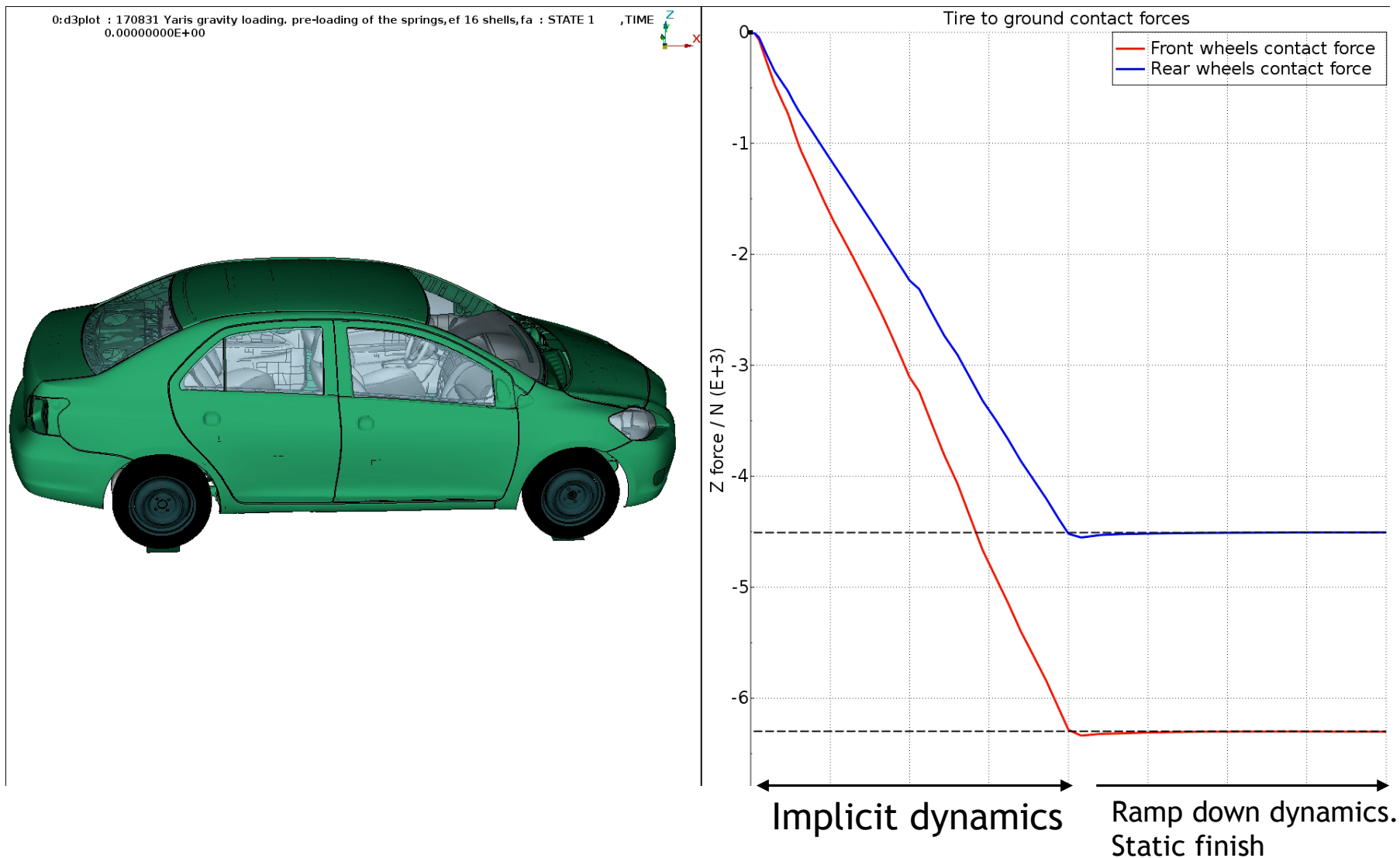
- The single surface contact was switched to Mortar contact (Note! Still one automatic single surface contact definition for the whole model)
- Added *IPBACK* to the tied contact for spot welds
- Switched to shell elform 16 using `*CONTROL_IMPLICIT_EIGENVALUE`
- The standard control card settings for non-linear implicit analyses from the Guideline were used
- The geometrical stiffness effect was disabled (*IGS* = 2 on `*CONTROL_IMPLICIT_GENERAL`)
- Rate effects were disabled (*IRATE* = 2 `*CONTROL_IMPLICIT_DYNAMICS`)

Studied load cases

- Gravity loading
 - Initially dynamic with static finish
- Door sag loading
 - Static load case
 - At 70° door opening angle
 - Gravity, and 1 kN loading at striker
- Hood / fender loading
 - Force controlled (1 kN), static load case
 - Fender/side position
 - Front position
- Door slam loading
 - Purely dynamic load cases
 - Prescribed initial rotational velocity of the door about the hinge: 1 rad/s, 2 rad/s

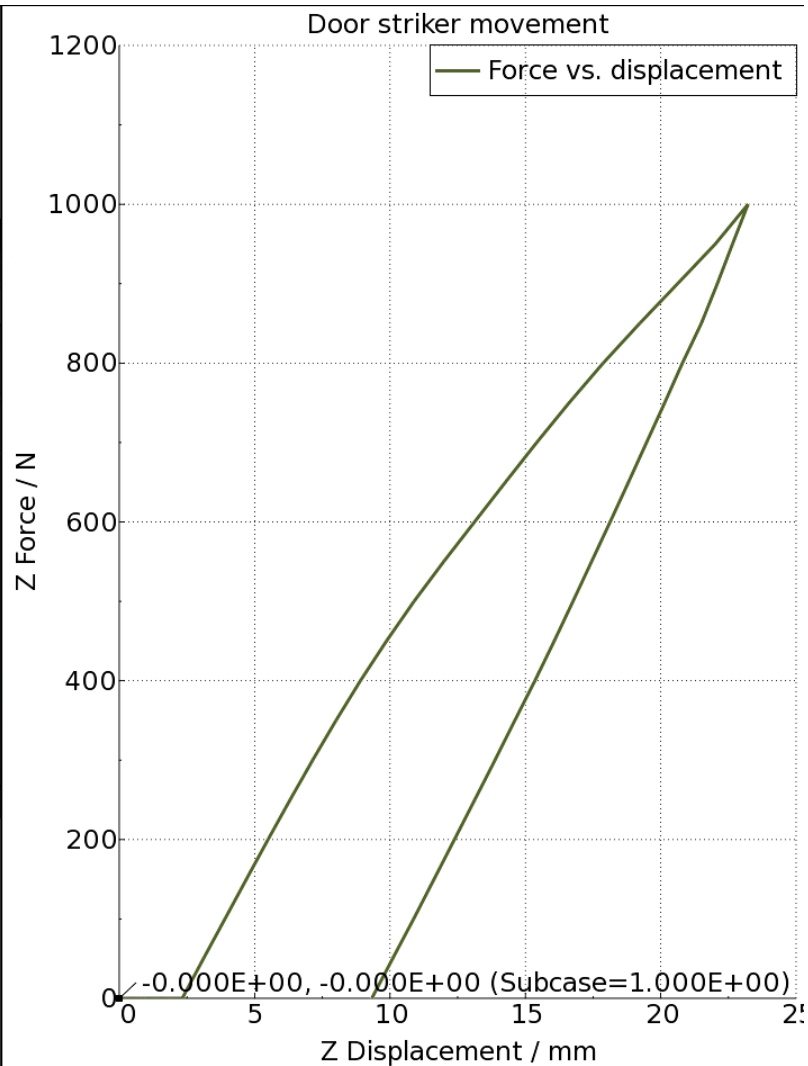
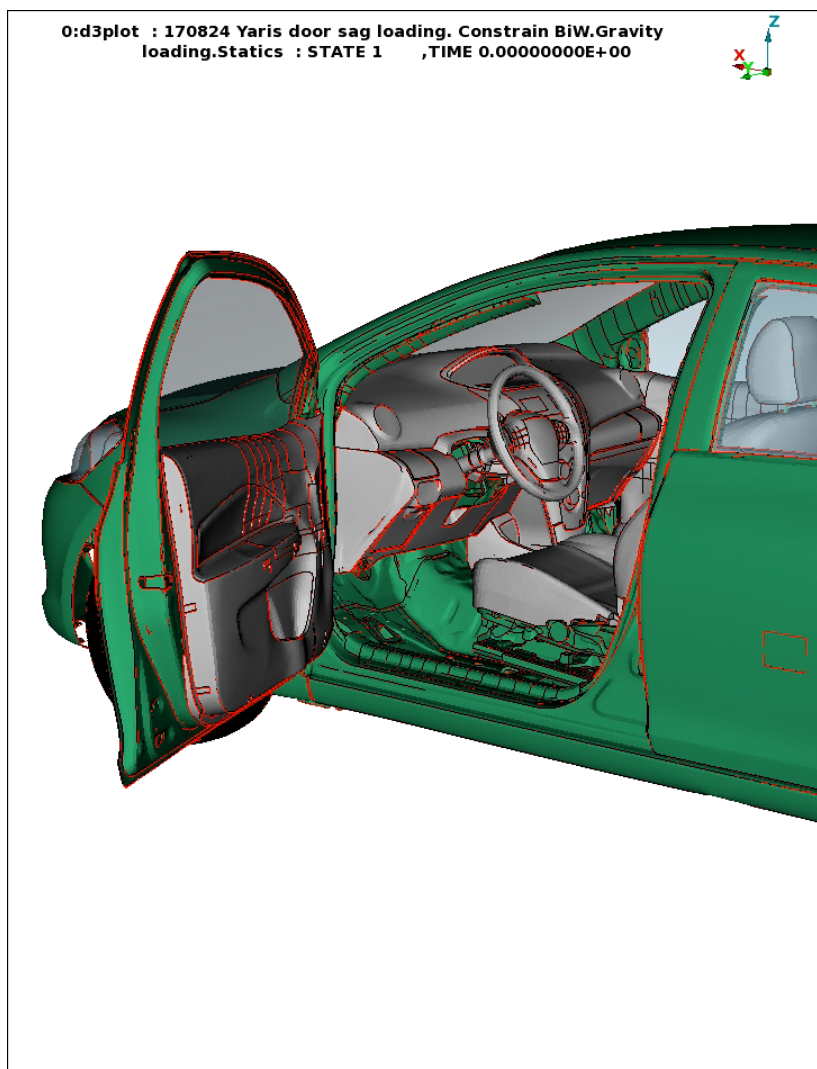
Run with
constrained
suspension

Gravity loading

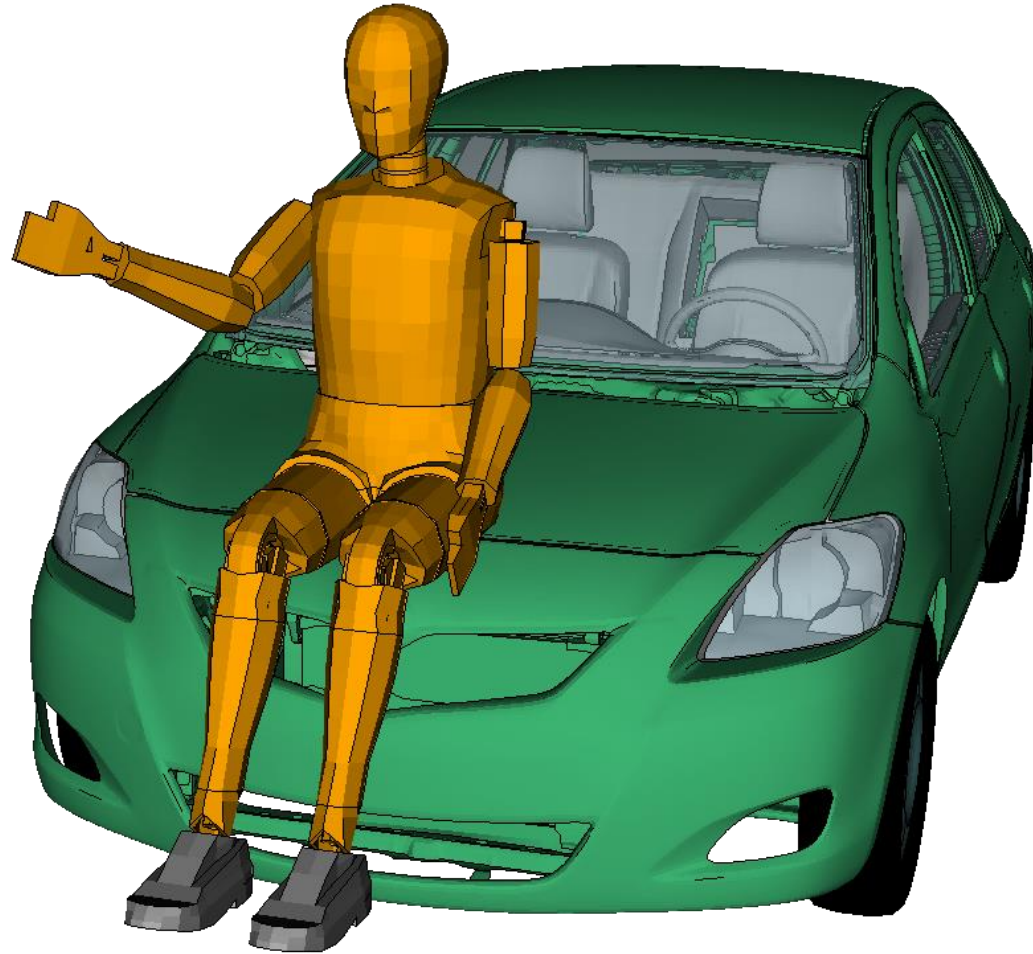


Door sag loading.

Solution time: 8h 48min on 16 cores

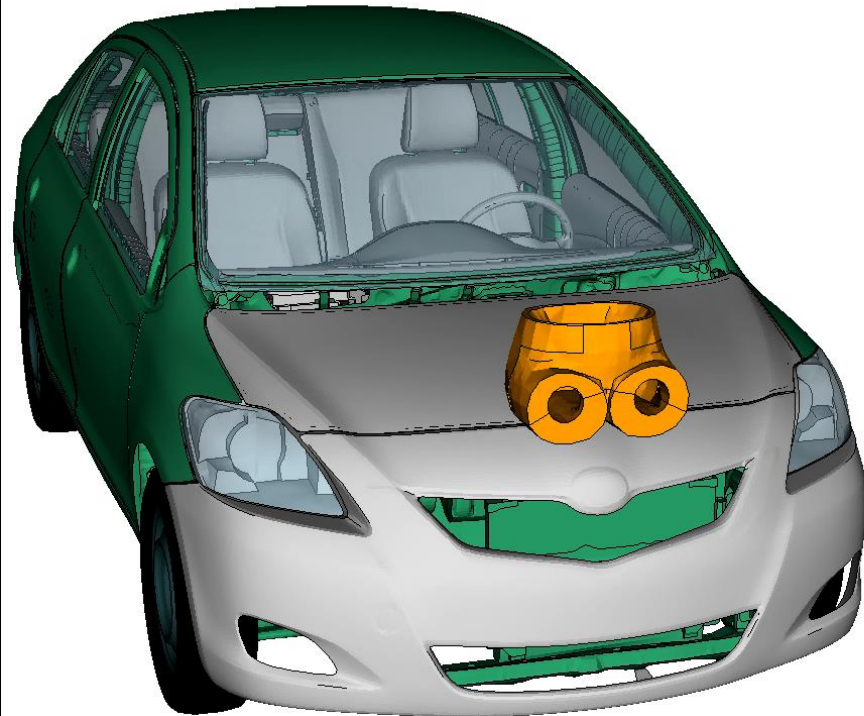


Hood / fender loading



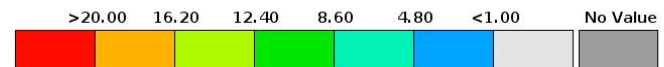
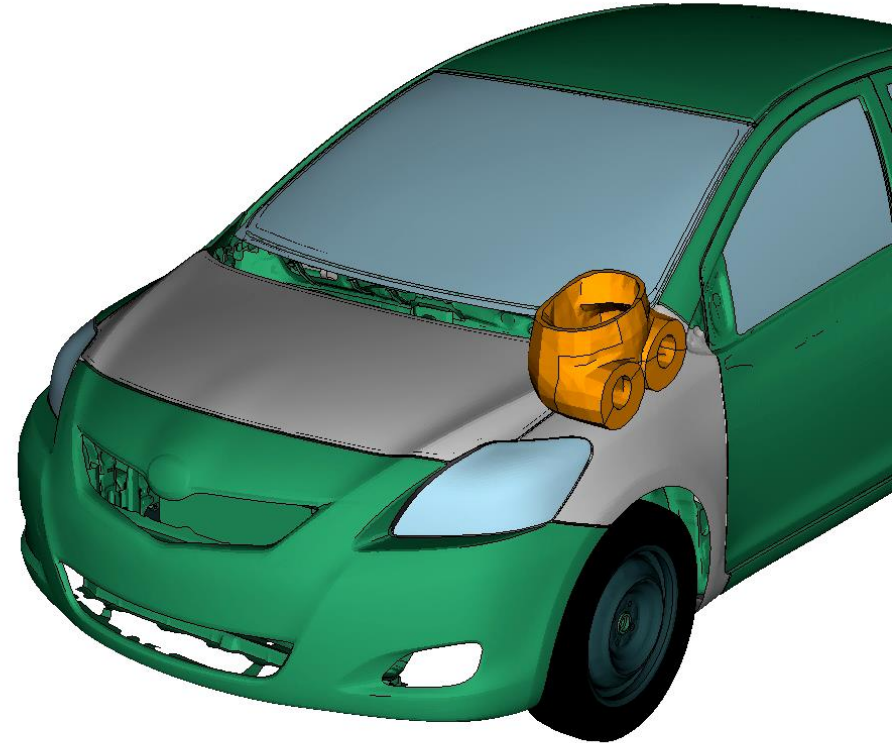
Hood / fender loading.

0:d3plot : 170908 Yaris Fender loading by prescribed force and unloading. Constrain : Scalar: : Magnitude of Displacements : : STATE 1 ,TIME 0.00000000E+00

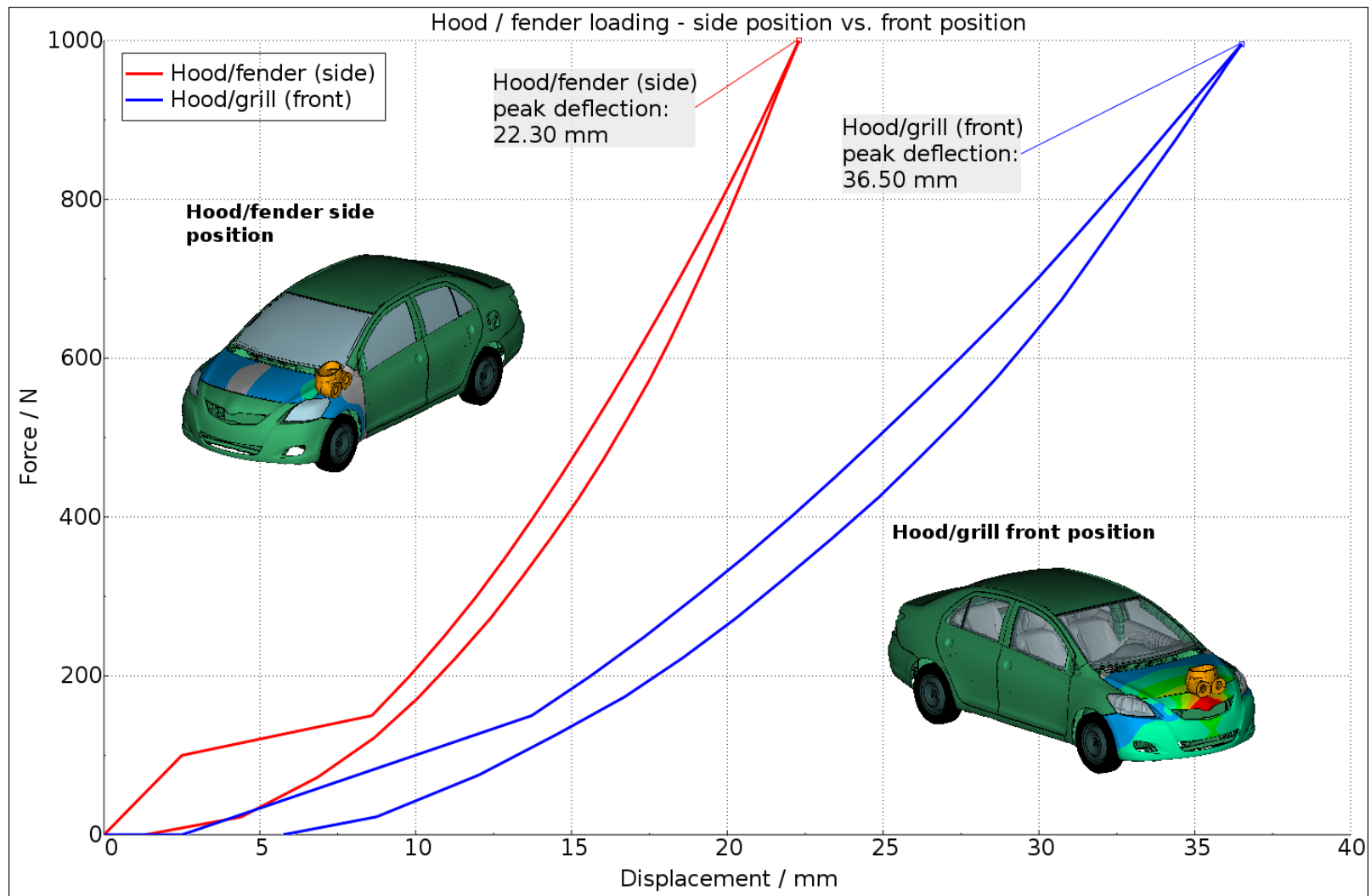


Solution time: 7h 38 min on 24 cores

1:d3plot : 170825 Yaris Fender loading by prescribed force and unloading. Constrain : Scalar: : Magnitude of Displacements : : STATE 1 ,TIME 0.00000000E+00



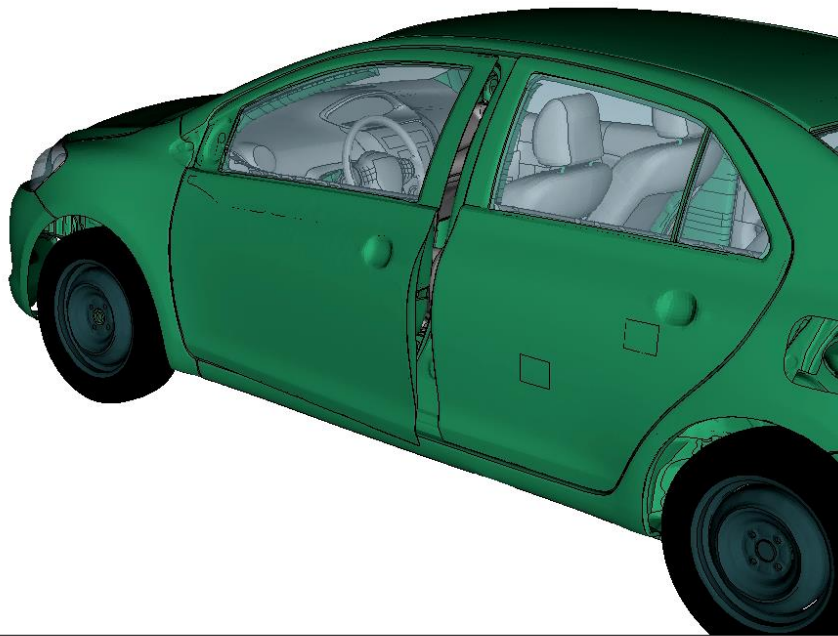
Hood / fender loading



Door slam.

Solution time: 11h 9min on 16 cores

Initial velocity: 2 rad / s



Initial velocity: 1 rad / s



Summary: From explicit to implicit

- It's possible to convert a large-scale explicit crash model to run in implicit by minimal modifications
 - Both static and dynamic load cases can be studied in implicit
- LS-DYNA Implicit works well
 - Also for large scale models, single surface contacts
- LSTC and Dynamore are continuously working on making implicit more useful and easy-to-use
 - LS-PrePost GUI for simplified set-up of implicit analyses
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N o r m a l t e r m i n a t i o n

Thank you!

